

# ME 476

## Solar Energy

### UNIT SIX

#### SOLAR ENERGY APPLICATIONS

#### *ABSORPTION REFRIGERATION*

# Absorption Refrigeration

- Absorption refrigeration systems belong to the class of vapor cycles similar to vapor compression refrigeration systems.
- However, unlike vapor compression refrigeration systems, the required input to absorption systems is in the form of heat.
- Hence these systems are also called as heat operated.
- Since these systems run on low-grade thermal energy, they are preferred when low-grade energy such as waste heat or solar energy is available.
- Since conventional absorption systems use natural refrigerants such as water or ammonia they are environment friendly.

## Advantages

- Natural, environmentally-friendly refrigerants (e.g. water, ammonia).
- Minimal use of electric power (heat-operated).

## Disadvantages

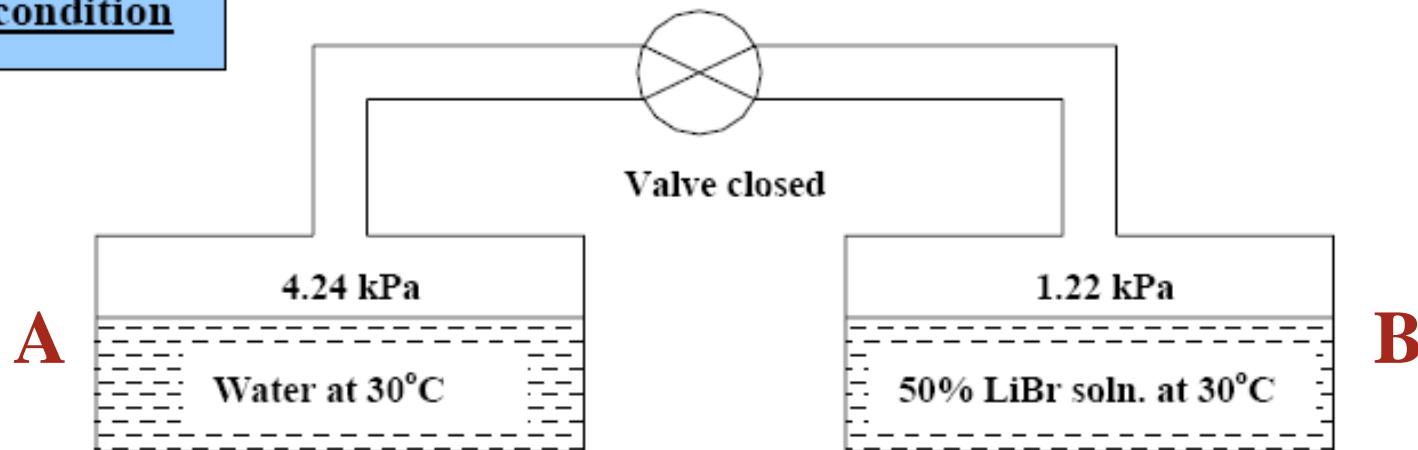
- Lower COP than vapor compression systems.
- Physically larger equipment.
- Higher capital cost.
- Possibility of corrosion problems.

Most commercially available absorption refrigeration systems use one of the two following refrigerant-absorbent pairs:

- Water (refrigerant) + Lithium-Bromide/Water Solution (absorbent)
- Ammonia (refrigerant) + Water/Ammonia Solution (absorbent)

- In the following example, we will consider an absorption system with water as the pure solvent and lithium bromide/water mixture as the solution.
- Consider two vessels, Vessel A containing pure solvent (water) and Vessel B containing a 50% LiBr/Water solution.

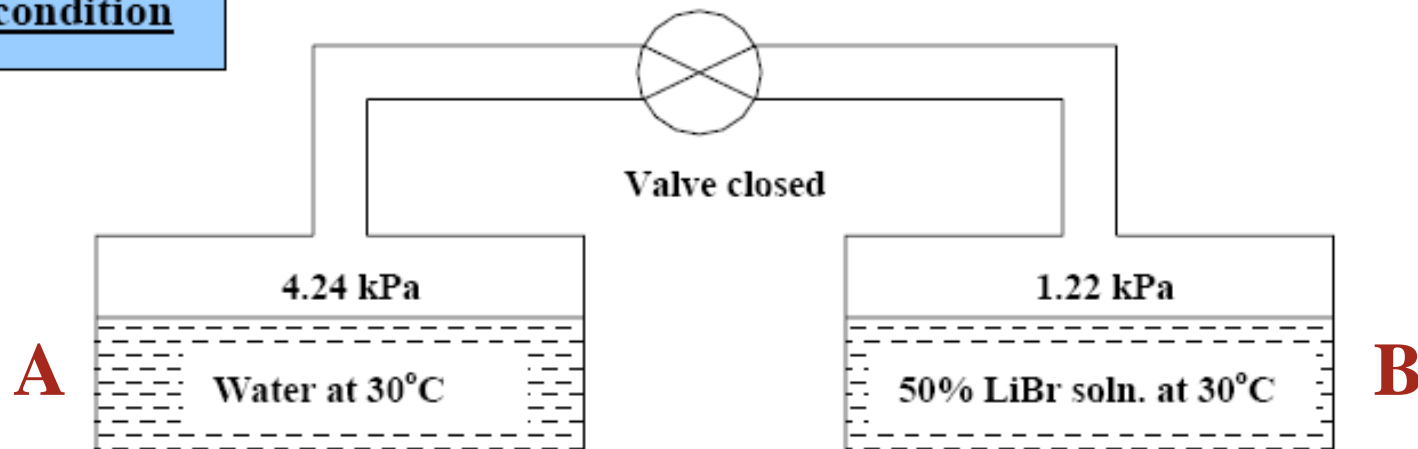
## a) Initial condition



# Basic Principles

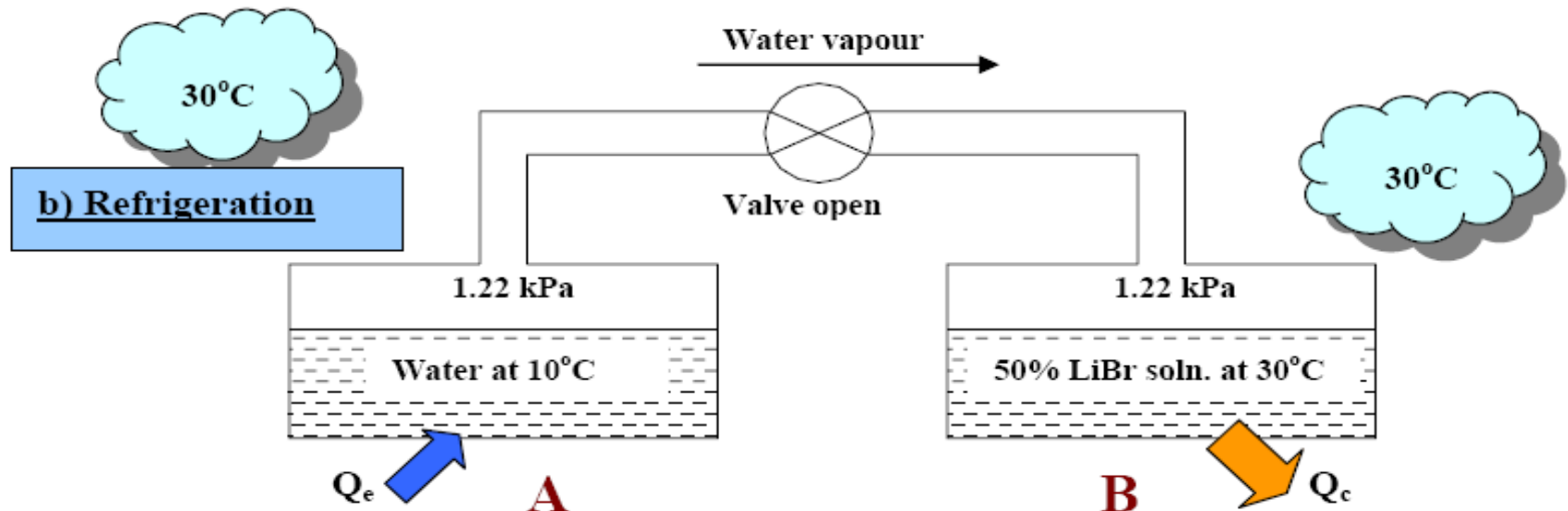
- Initially, the two vessels are disconnected and the temperature is the same in both vessels.
- In this case, the vapor pressure of water in Vessel A is greater than in Vessel B.
- This is a natural characteristic of the two substances.

## a) Initial condition



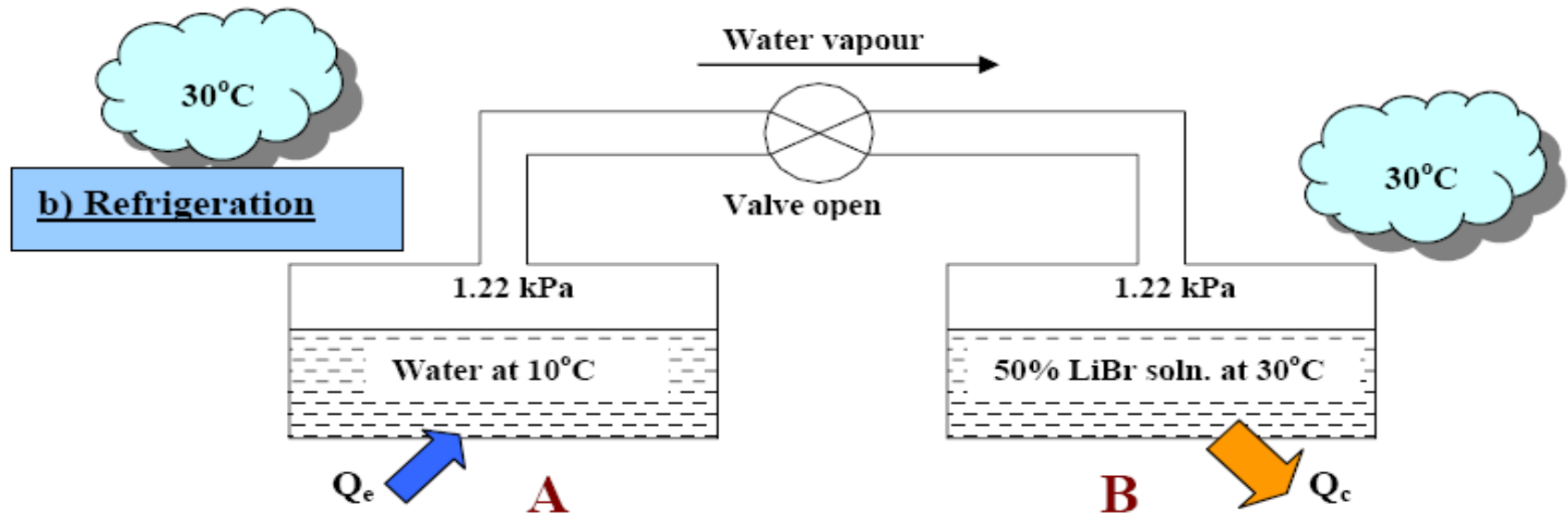
# Basic Principles

- If the two vessels are connected, and the vapor pressure of the LiBr/water solution needs to be maintained, the temperature of the pure solvent will have to drop.
- If the solution is at ambient temperature, the pure solvent (water) will be at a temperature lower than the ambient.
- Hence refrigeration effect is produced at Vessel A due to this temperature difference.



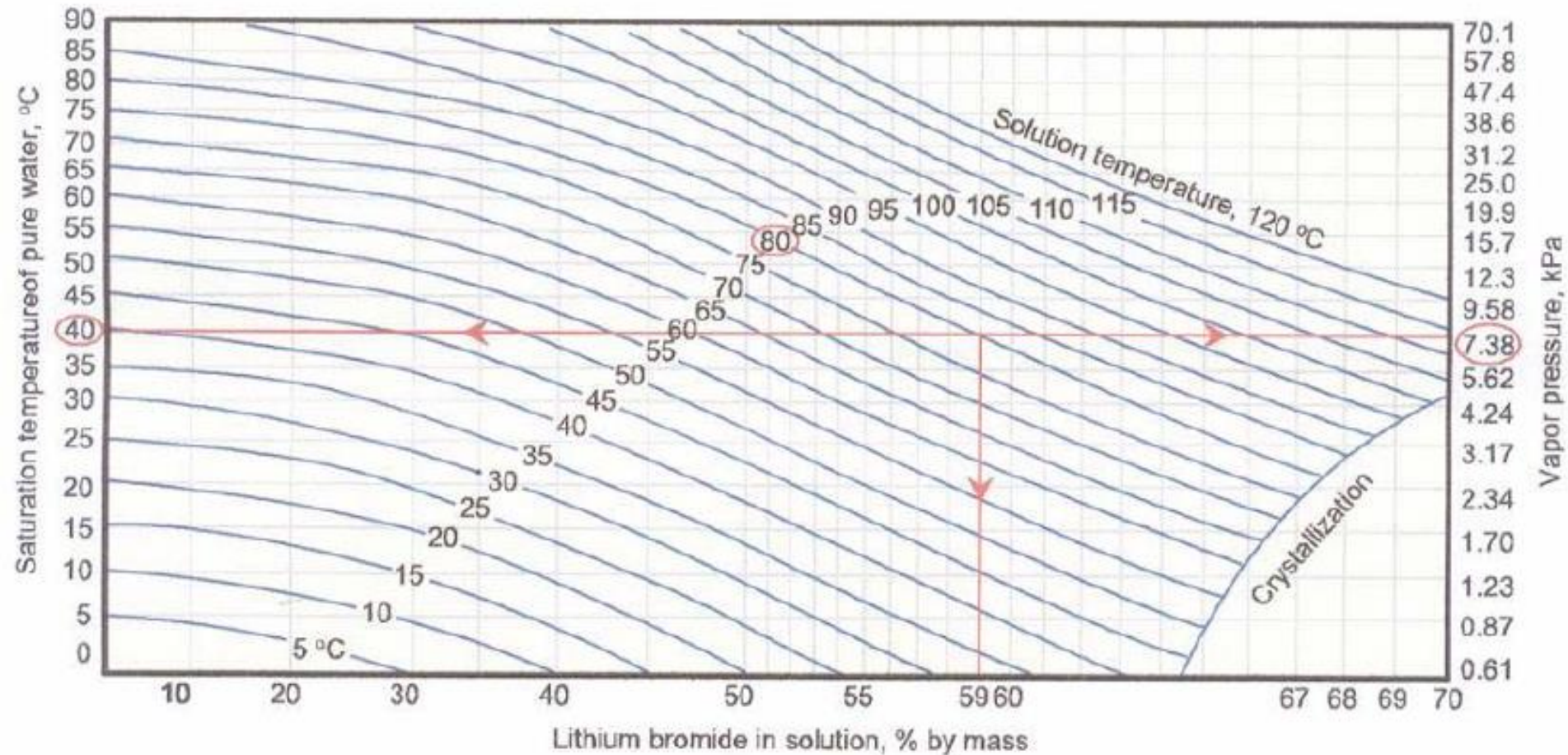
# Basic Principles

- The pure solvent (water) absorbs heat from the surroundings, evaporates, flows to Vessel B, and is absorbed by the solution.
- To maintain the temperature of Vessel B, heat must be rejected





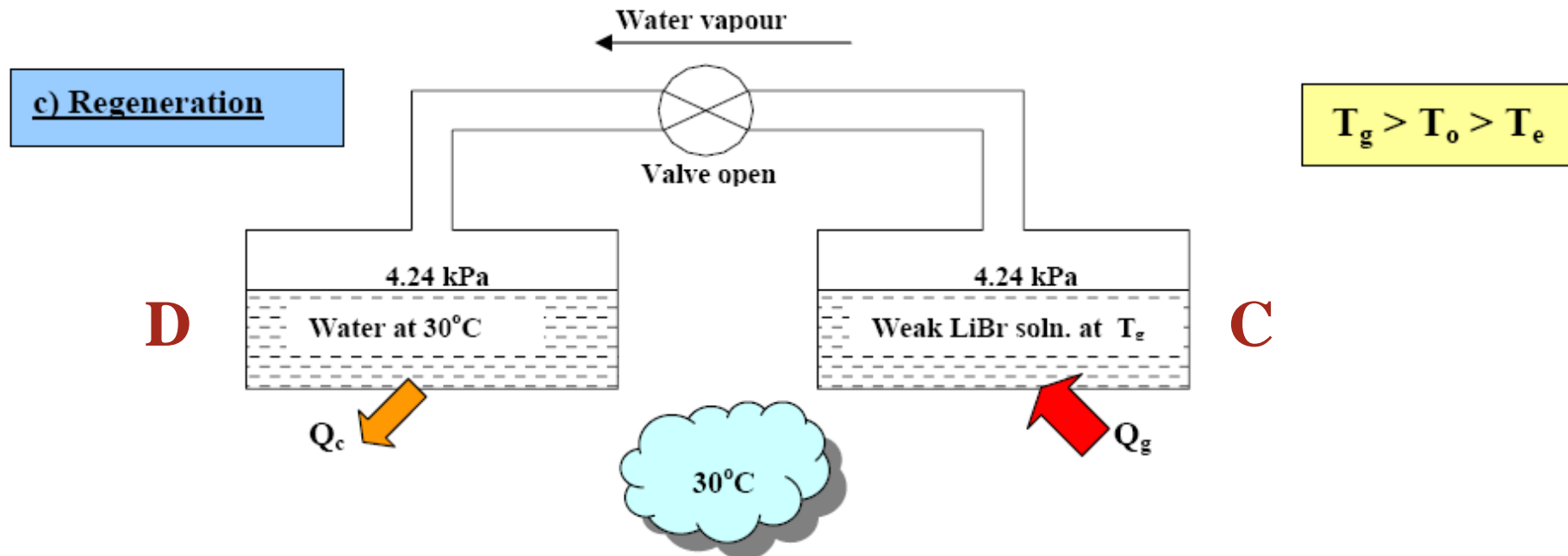
# Pressure-Temperature-Concentration Diagram



Concentration (x) is defined as: 
$$x = \frac{m_{\text{LiBr}}}{m_{\text{LiBr}} + m_{\text{water}}}$$

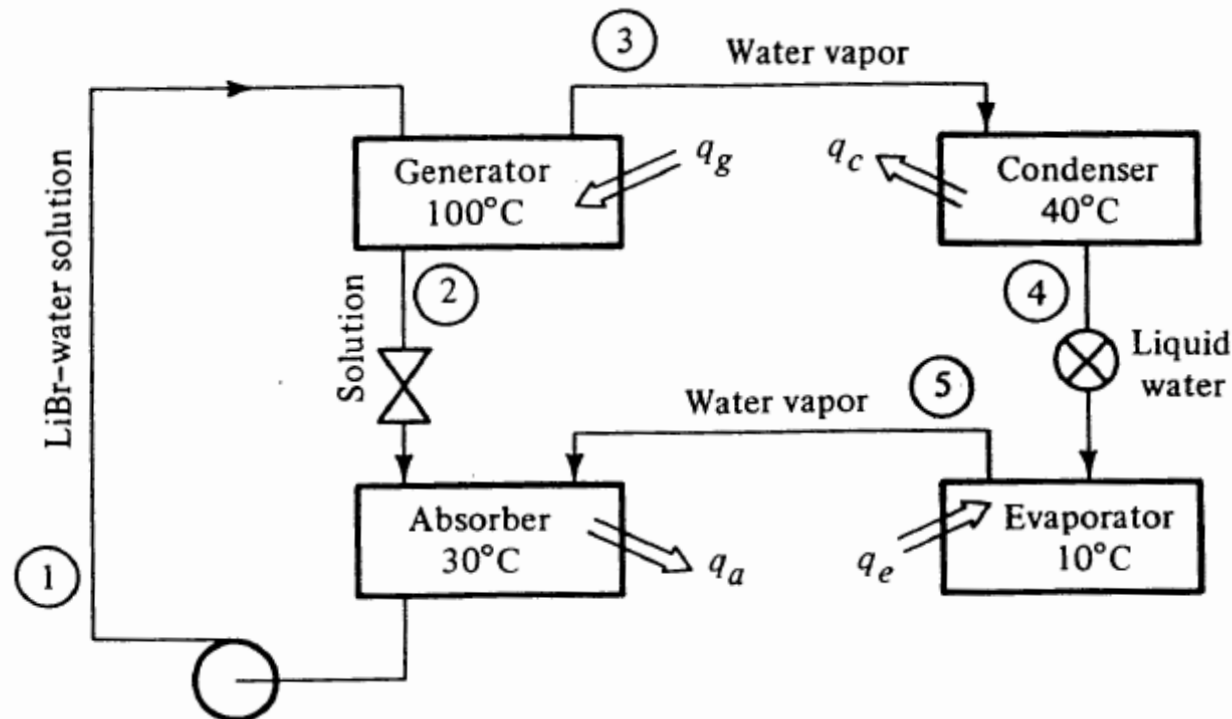
# Basic Principles

- On the other hand, consider Vessel C containing a LiBr/water solution and Vessel D containing pure solvent (water).
- Vessel C receives heat from an external source.
- LiBr/water solution drives off some of the water vapor.
- This water vapor passes to Vessel D.
- In Vessel D, the water vapor is condensed at a lower temperature by rejecting heat.



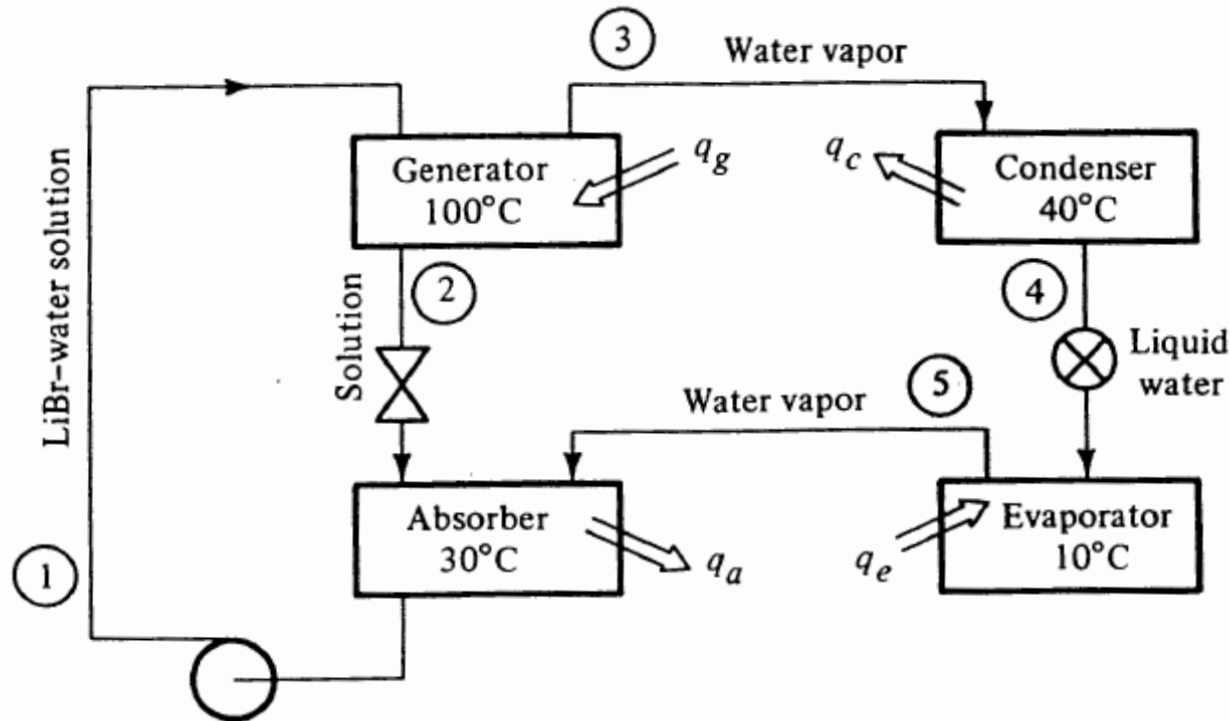
# Absorption Refrigeration System

- Low-pressure vapor from the evaporator is absorbed by the liquid solution in the absorber.
- To maintain the absorber temperature, heat must be rejected.
- The pump receives low-pressure liquid from absorber, elevates its pressure, and delivers it to the generator.



# Absorption Refrigeration System

- In the generator, heat from a high-temperature source drives off the vapor that had been absorbed by solution.
- The vapor enters the condenser while the liquid solution returns to the absorber through a throttling valve.



The general definition of the coefficient of performance is:

$$\text{COP} = \frac{\text{desired output}}{\text{required input}}$$

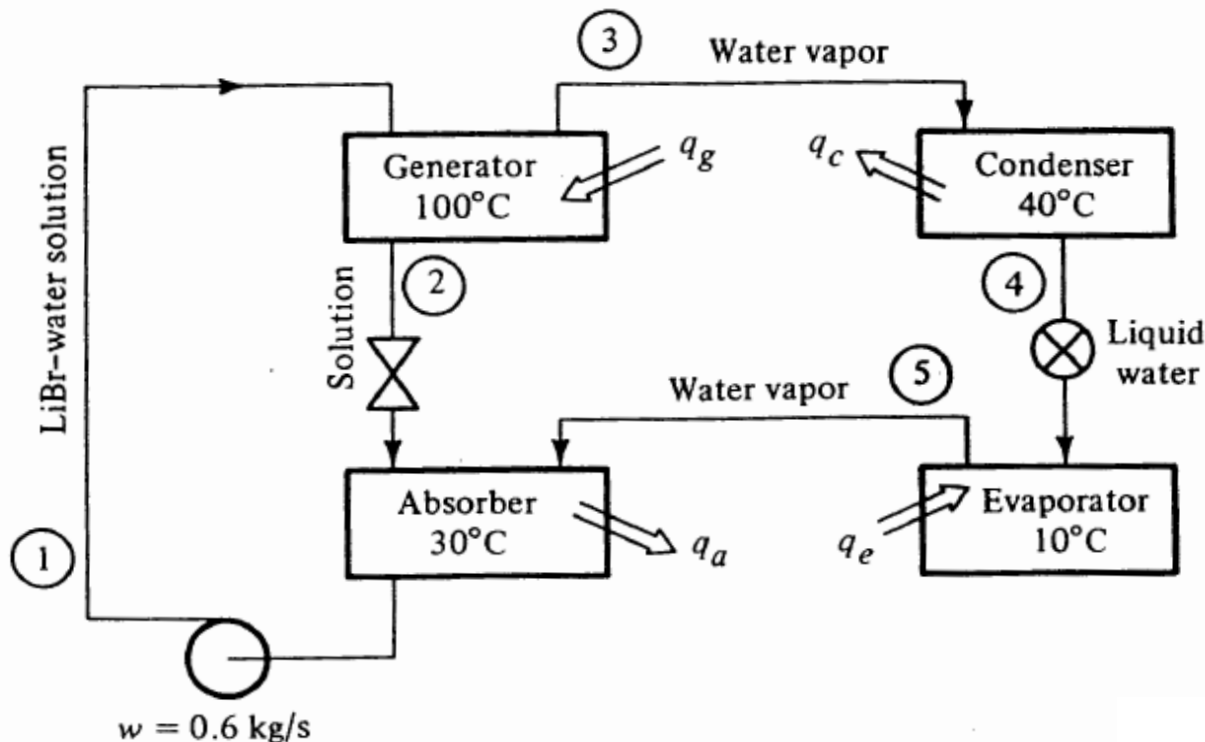
Therefore, in the case of absorption refrigeration, the COP is:

$$\text{COP}_{\text{abs}} = \frac{\text{refrigeration rate}}{\text{rate of heat addition at generator}}$$

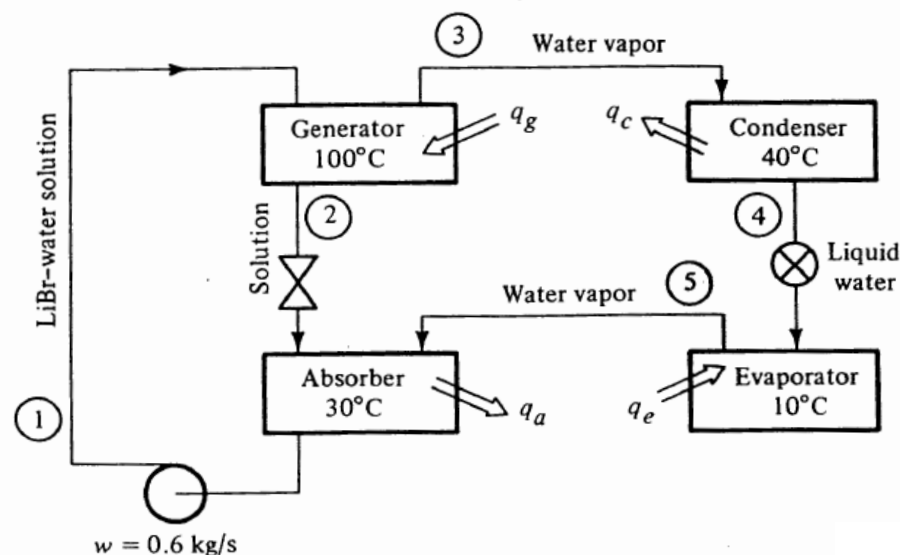
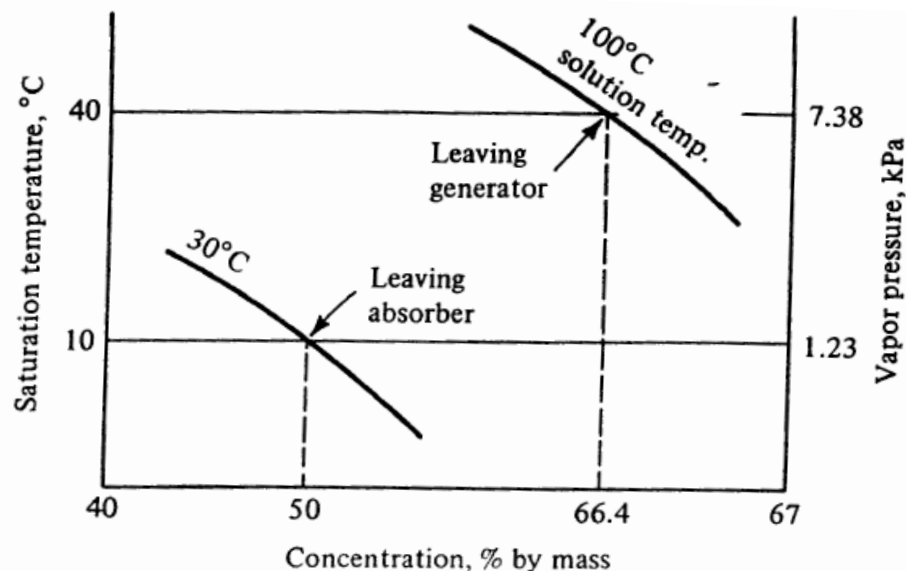
The power consumed by the pump is not included in the COP expression because it is too small compared to the heat input.

## Example

Compute the mass flow rate of refrigerant (water) through the condenser and evaporator in the cycle shown below if the pump delivers 0.6 kg/s and the following temperatures prevail: generator, 100°C; condenser, 40°C; evaporator, 10°C; and absorber, 30°C.



# Calculation of Mass Flow Rates



Total mass-flow balance:  $w_2 + w_3 = w_1 = 0.6$

LiBr balance:  $w_1 x_1 = w_2 x_2$   
 $0.6(0.50) = w_2(0.664)$

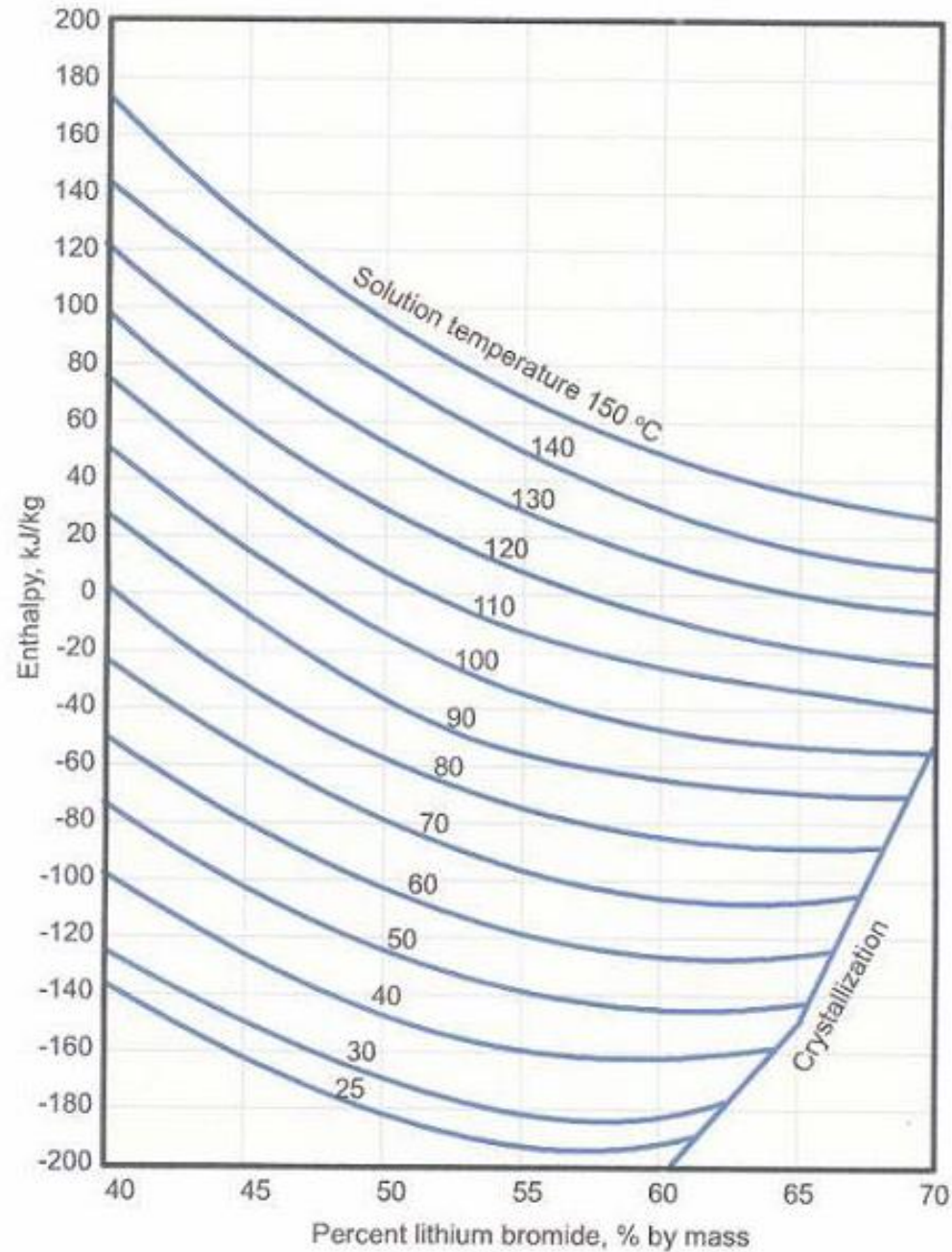
Solving the two balance equations simultaneously gives

$$w_2 = 0.452 \text{ kg/s} \quad \text{and} \quad w_3 = 0.148 \text{ kg/s}$$

Approximately 4 kg of solution is pumped for each kilogram of refrigerant water vapor developed.



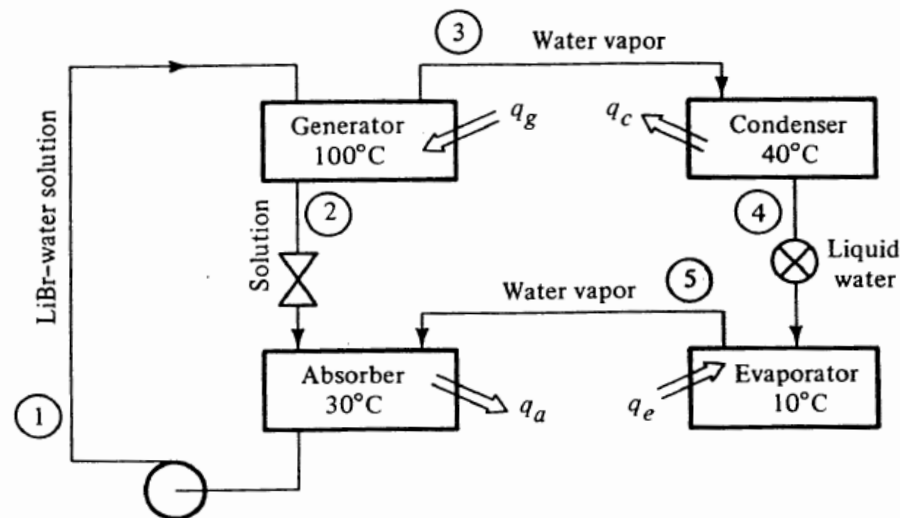
# Enthalpy of LiBr-Water Solution





## Example

For the absorption system shown, compute  $q_g$ ,  $q_a$ ,  $q_c$ ,  $q_e$ , and the COP.



**Solution** The flow rates and solution concentrations have already been determined in Example 17-2:  $w_1 = 0.6$  kg/s,  $w_2 = 0.452$  kg/s, and  $w_3 = w_4 = w_5 = 0.148$  kg/s;  $x_1 = 50\%$ , and  $x_2 = 66.4\%$ .

The enthalpies of the solution can be read off Fig. 17-8:

$$h_1 = h \text{ at } 30^\circ\text{C and } x \text{ of } 50\% = -168 \text{ kJ/kg}$$

$$h_2 = h \text{ at } 100^\circ\text{C and } x \text{ of } 66.4\% = -52 \text{ kJ/kg}$$

The enthalpies of water liquid and vapor are found from Table A-1:

$$h_3 = h \text{ of saturated vapor at } 100^\circ\text{C} = 2676.0 \text{ kJ/kg}$$

$$h_4 = h \text{ of saturated liquid at } 40^\circ\text{C} = 167.5 \text{ kJ/kg}$$

$$h_5 = h \text{ of saturated vapor at } 10^\circ\text{C} = 2520.0 \text{ kJ/kg}$$

# Energy Balance

The rates of heat transfer at each of the components can now be computed from energy balances:

$$\begin{aligned}q_g &= w_3 h_3 + w_2 h_2 - w_1 h_1 \\&= 0.148(2676) + 0.452(-52) - 0.6(-168) = 473.3 \text{ kW}\end{aligned}$$

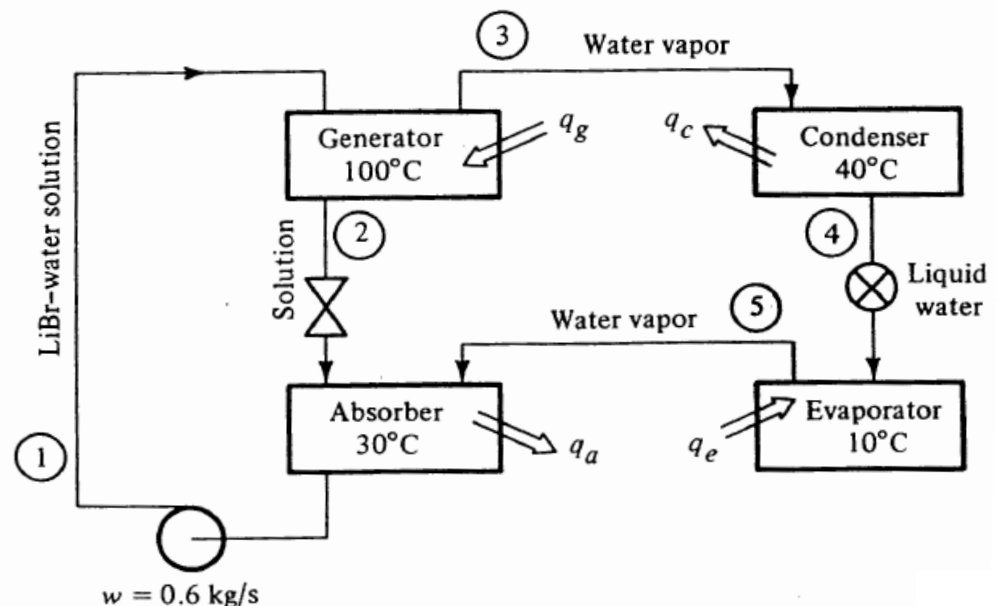
$$q_c = w_c h_3 - w_4 h_4 = 0.148(2676 - 167.5) = 371.2 \text{ kW}$$

$$\begin{aligned}q_a &= w_2 h_2 + w_5 h_5 - w_1 h_1 \\&= 0.452(-52) + 0.148(2520) - 0.6(-168) = 450.3 \text{ kW}\end{aligned}$$

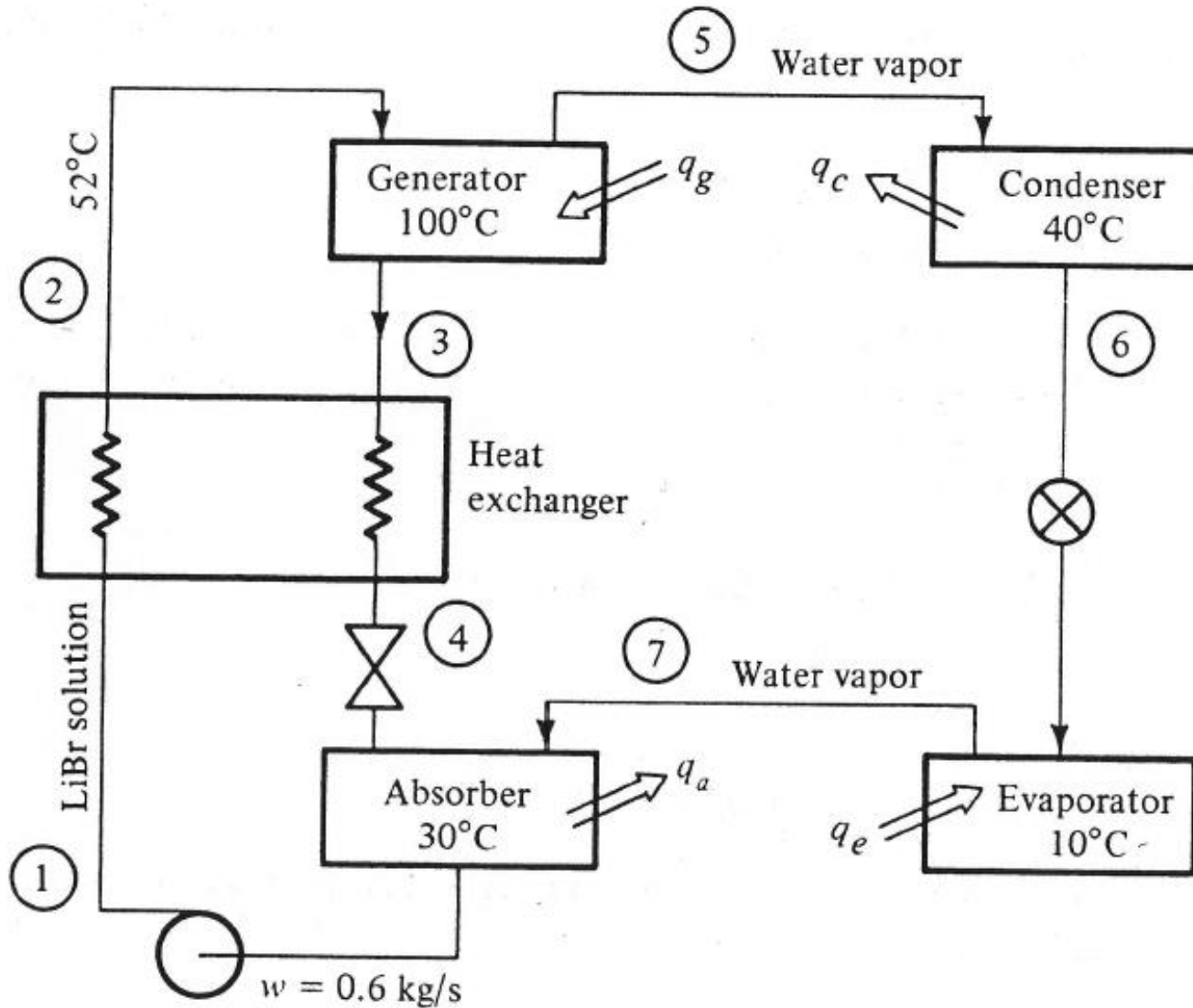
$$q_e = w_5 h_5 - w_4 h_4 = 0.148(2520 - 167.5) = 348.2 \text{ kW}$$

Finally,

$$\text{COP}_{\text{abs}} = \frac{q_e}{q_g} = \frac{348.2}{476.6} = 0.736$$



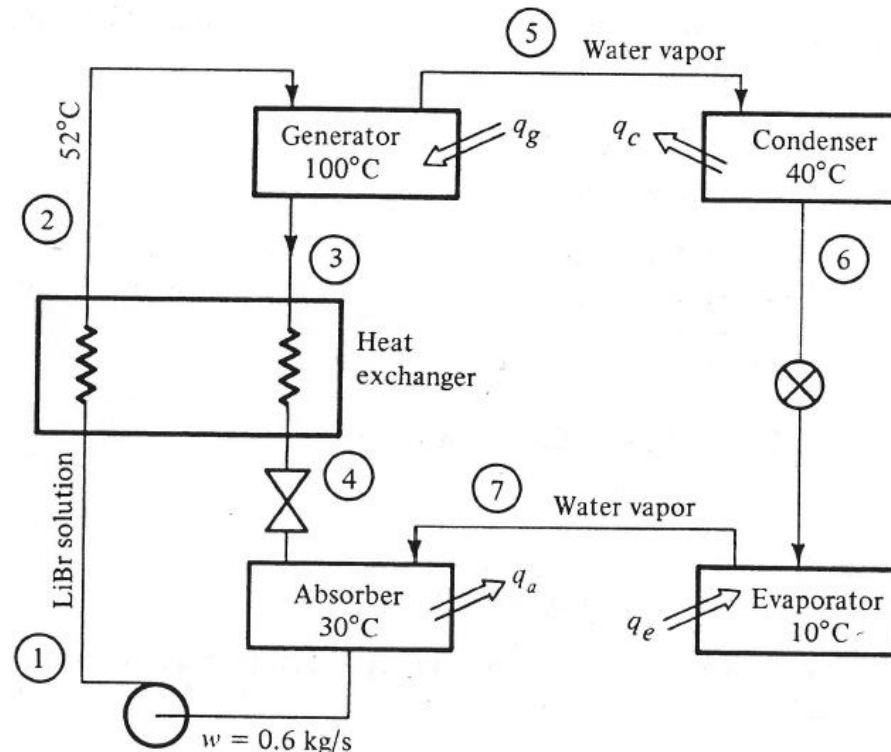
# Absorption Cycle with Heat Exchanger



# Absorption Cycle with Heat Exchanger

## Example

The simple cycle is modified by the insertion of a heat exchanger such that the temperature at Point 2 is  $52^{\circ}\text{C}$ . The mass flow rate delivered by the solution pump is  $0.6 \text{ kg/s}$ . What are the rates of energy transfer at each of the components and the  $\text{COP}_{\text{abs}}$  of this cycle?



# Absorption Cycle with Heat Exchanger

$$w_1 = w_2 = 0.6 \text{ kg/s} \quad w_3 = w_4 = 0.452 \text{ kg/s}$$

and

$$w_5 = w_6 = w_7 = 0.148 \text{ kg/s}$$

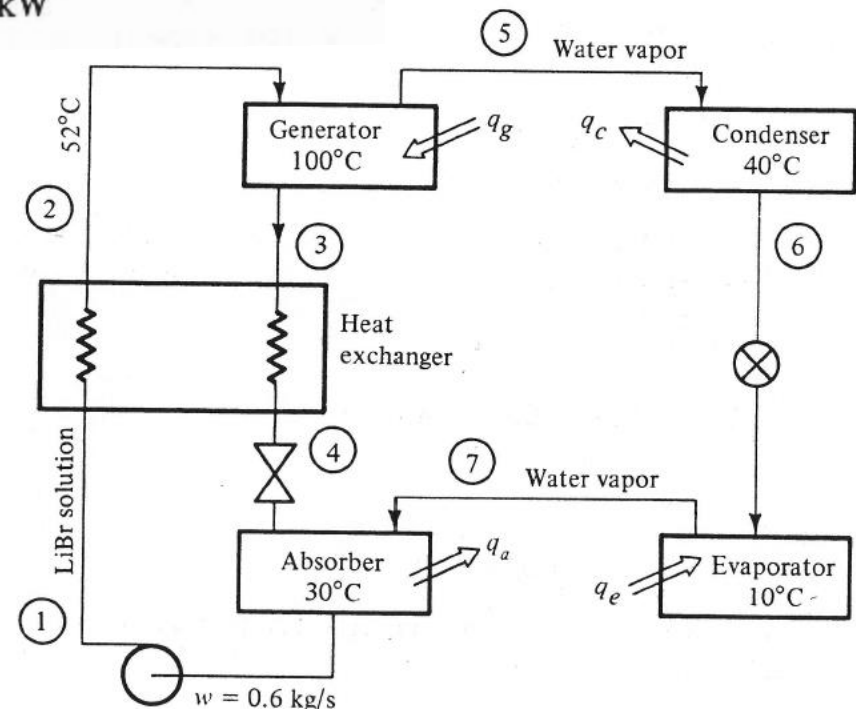
The enthalpies that remain unchanged are

$$h_1 = -168 \text{ kJ/kg} \quad h_3 = -52 \text{ kJ/kg}$$

$$h_5 = 2676.0 \text{ kJ/kg} \quad h_6 = 167.5 \text{ kJ/kg} \quad h_7 = 2520.0 \text{ kJ/kg}$$

The heat-transfer rates at the condenser and evaporator remain unchanged

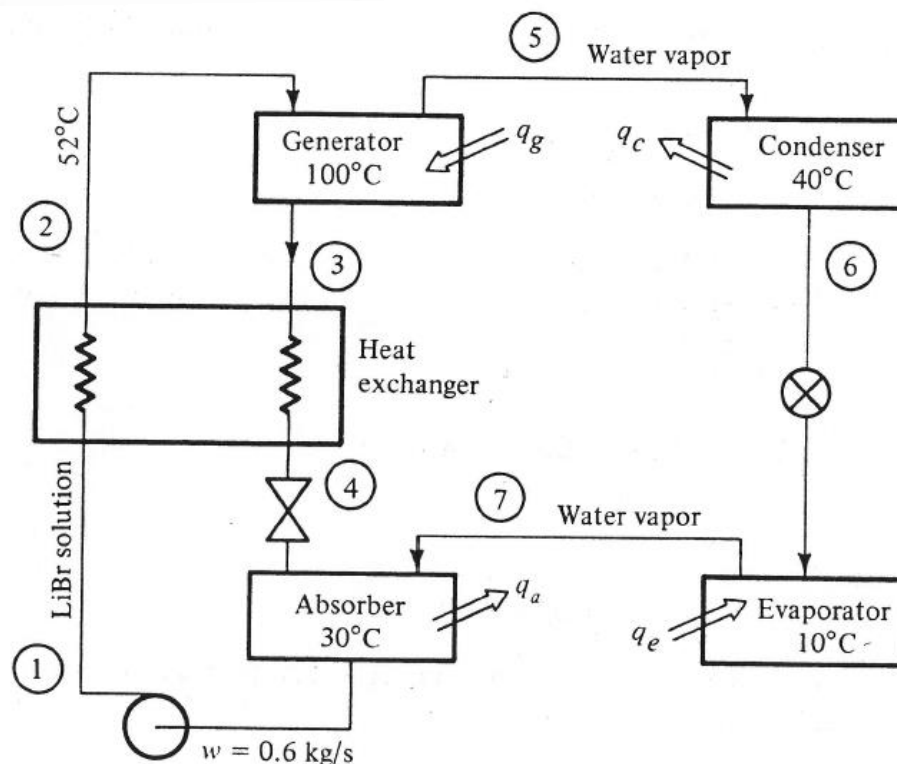
$$q_c = 371.2 \text{ kW} \quad \text{and} \quad q_e = 348.2 \text{ kW}$$



# Absorption Cycle with Heat Exchanger

The temperature of the 50% solution leaving the heat exchanger at point 2 is 52°C, and the solution at that condition has an enthalpy of -120 kJ/kg, as indicated by Fig. 17-8. The rate of heat absorbed by the solution passing from the absorber to the generator  $q_{hx}$  is

$$q_{hx} = w_1(h_2 - h_1) = 0.6[-120 - (-168)] = 28.8 \text{ kW}$$



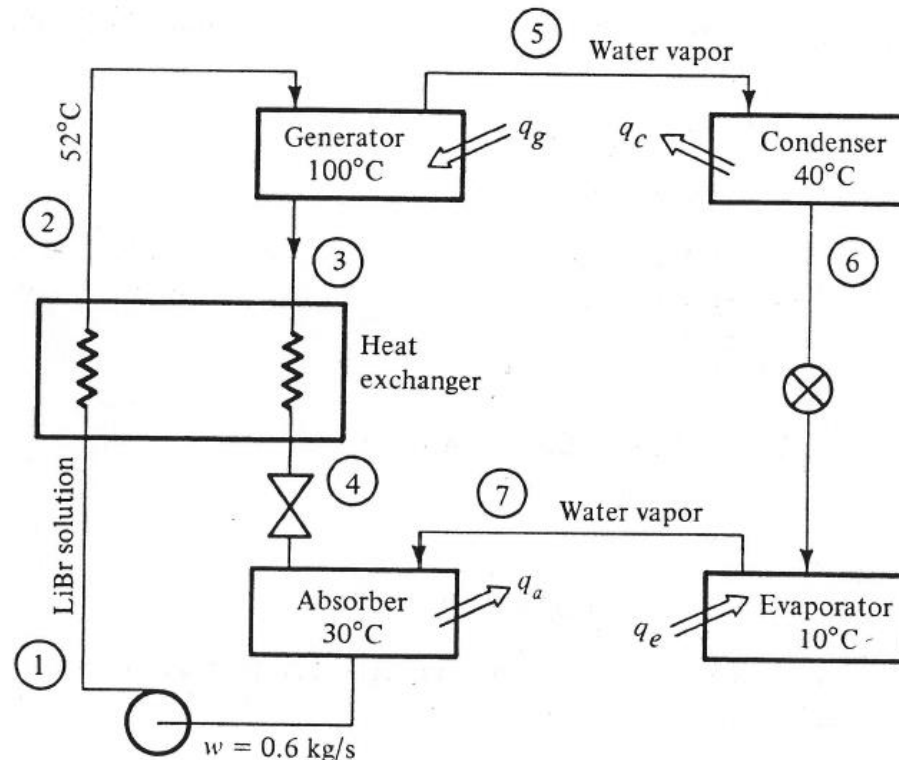


# Absorption Cycle with Heat Exchanger

$$q_{hx} = 28.8 \text{ kW} = w_3(h_3 - h_4) = 0.452(-52 - h_4)$$

$$h_4 = -116 \text{ kJ/kg}$$

$$\begin{aligned} q_g &= w_5 h_5 + w_3 h_3 - w_2 h_2 \\ &= 0.148(2676.0) + 0.452(-52) - 0.6(-120) = 444.5 \text{ kW} \end{aligned}$$

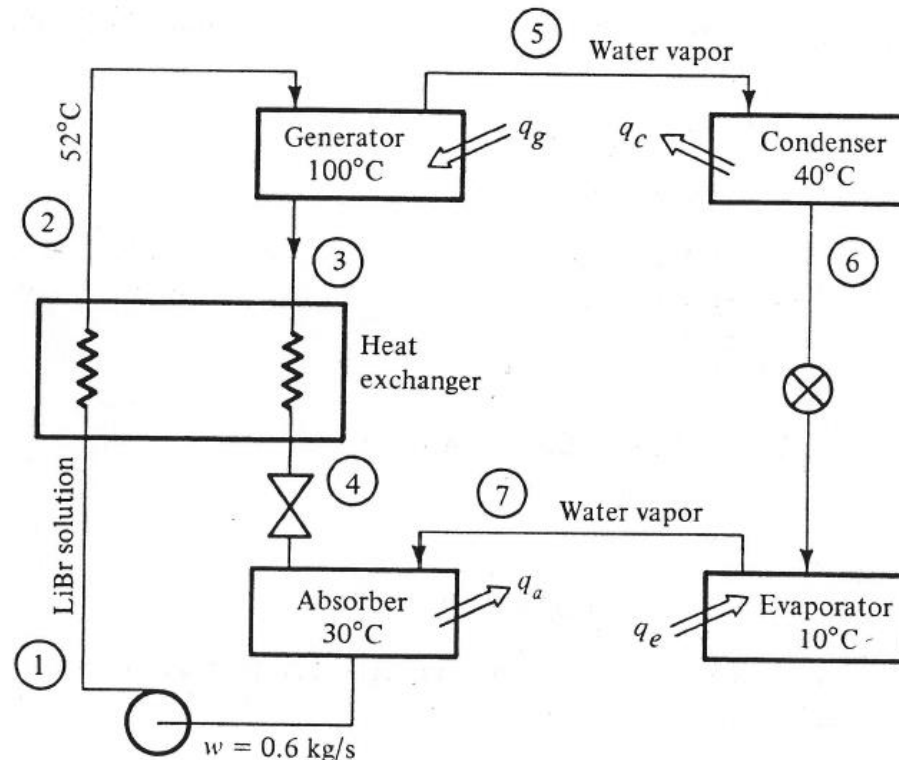


# Absorption Cycle with Heat Exchanger

$$q_a = w_7 h_7 + w_4 h_4 - w_1 h_1$$

$$= 0.148(2520) + 0.452(-116) - 0.6(-168) = 421.3 \text{ kW}$$

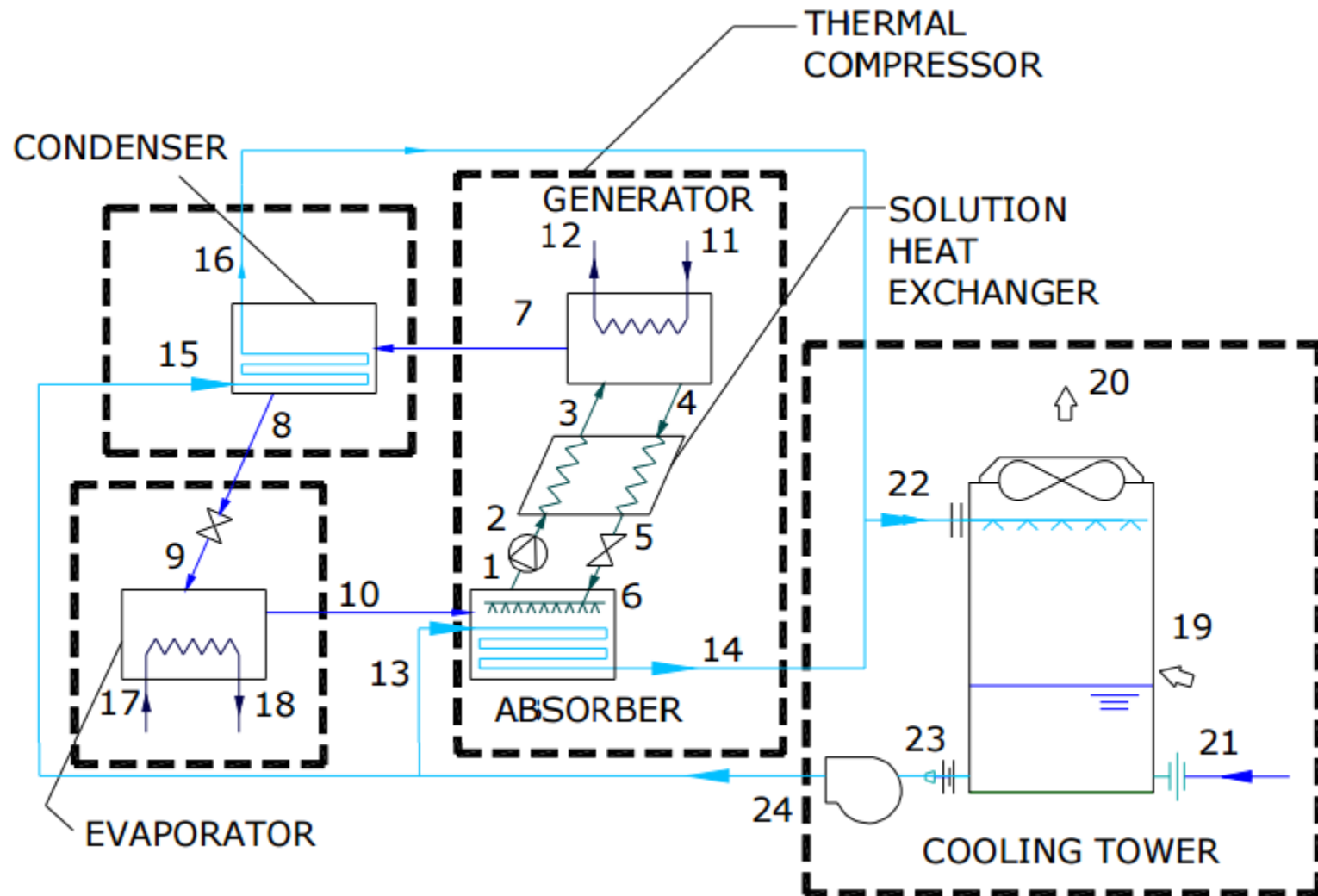
$$\text{COP}_{\text{abs}} = \frac{q_e}{q_g} = \frac{348.2}{444.5} = 0.783$$





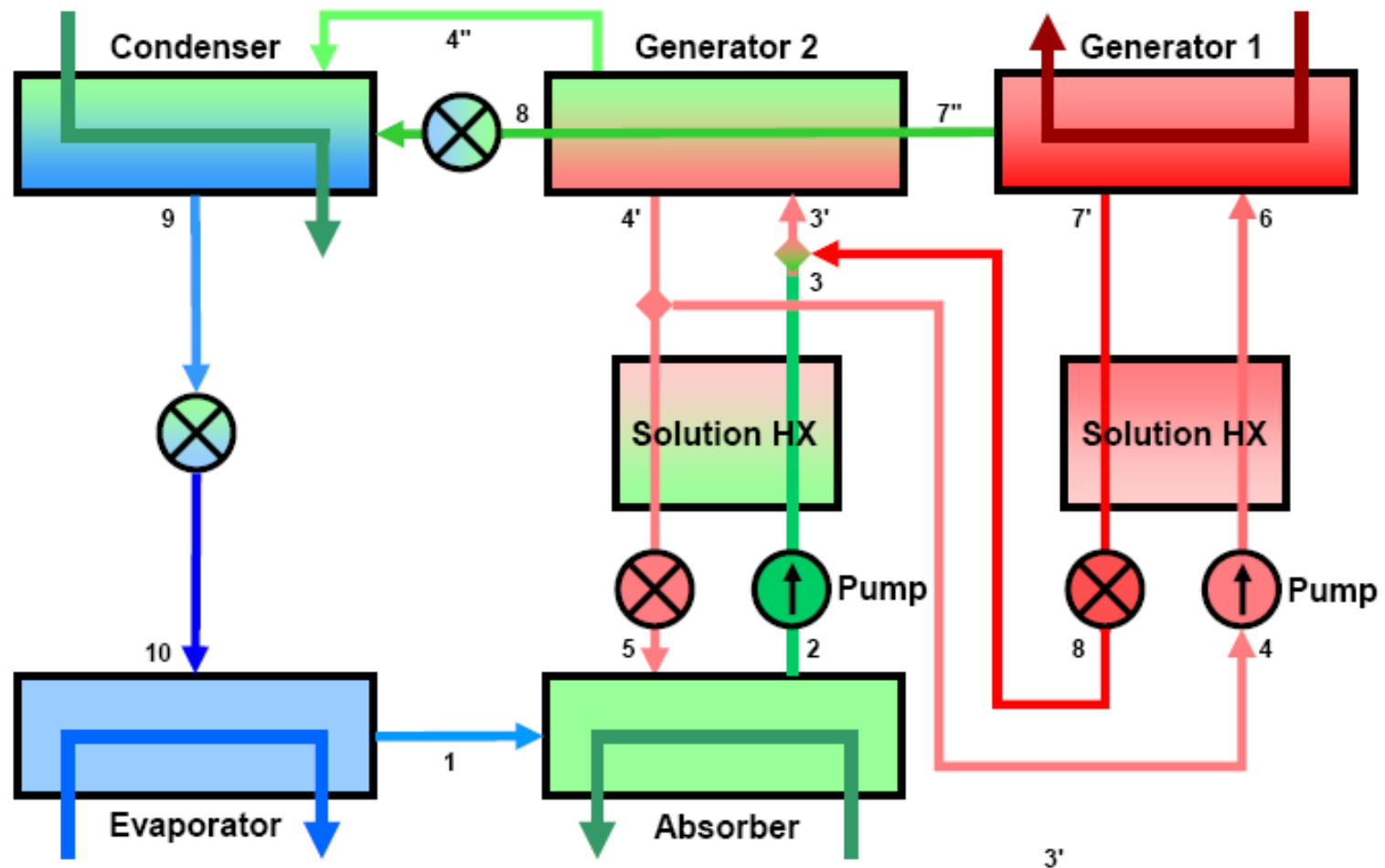
# Single Effect Absorption Cycle

## Configuration of a Typical Commercial System

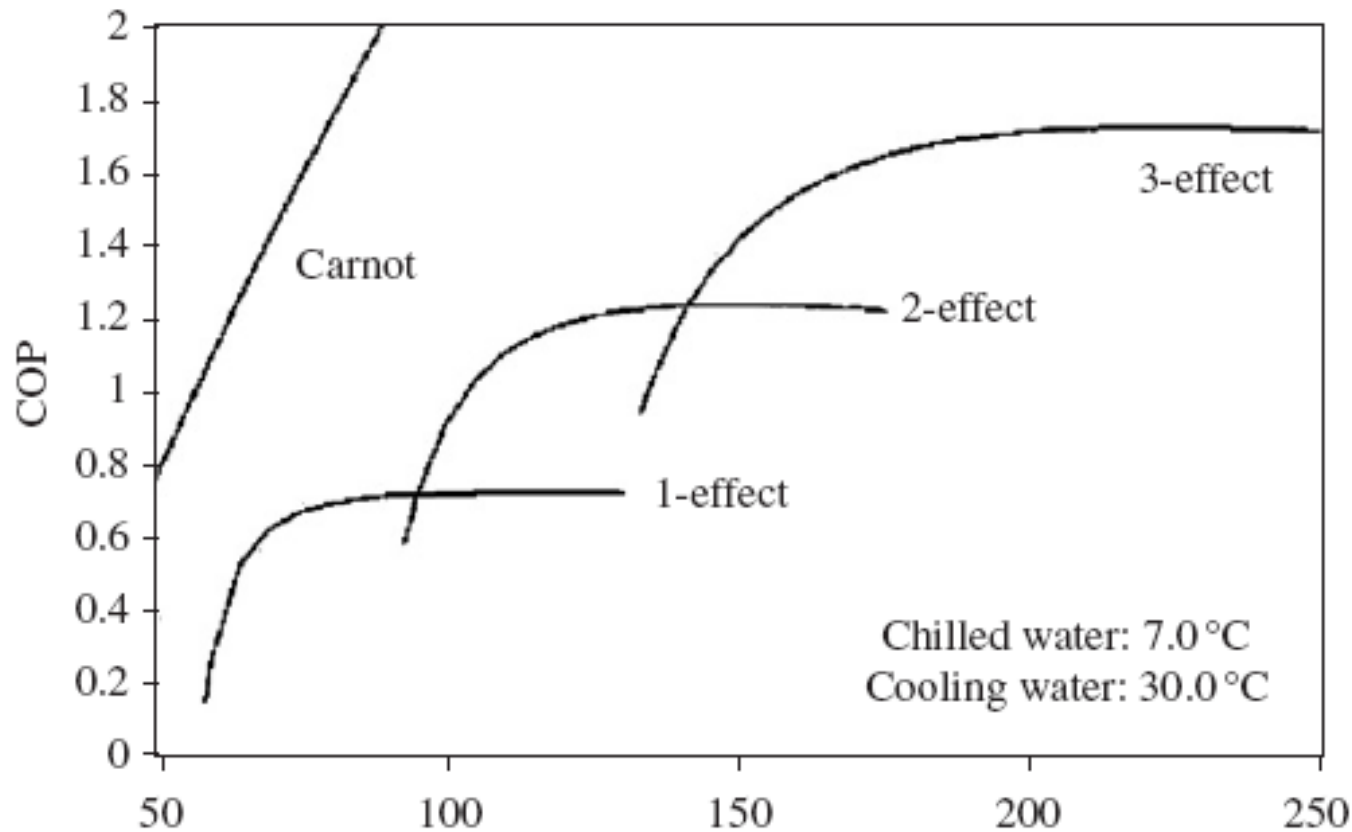


# Double Effect Absorption Cycle

## Configuration of a Typical Commercial System



# Performance Comparison



# **ME 476**

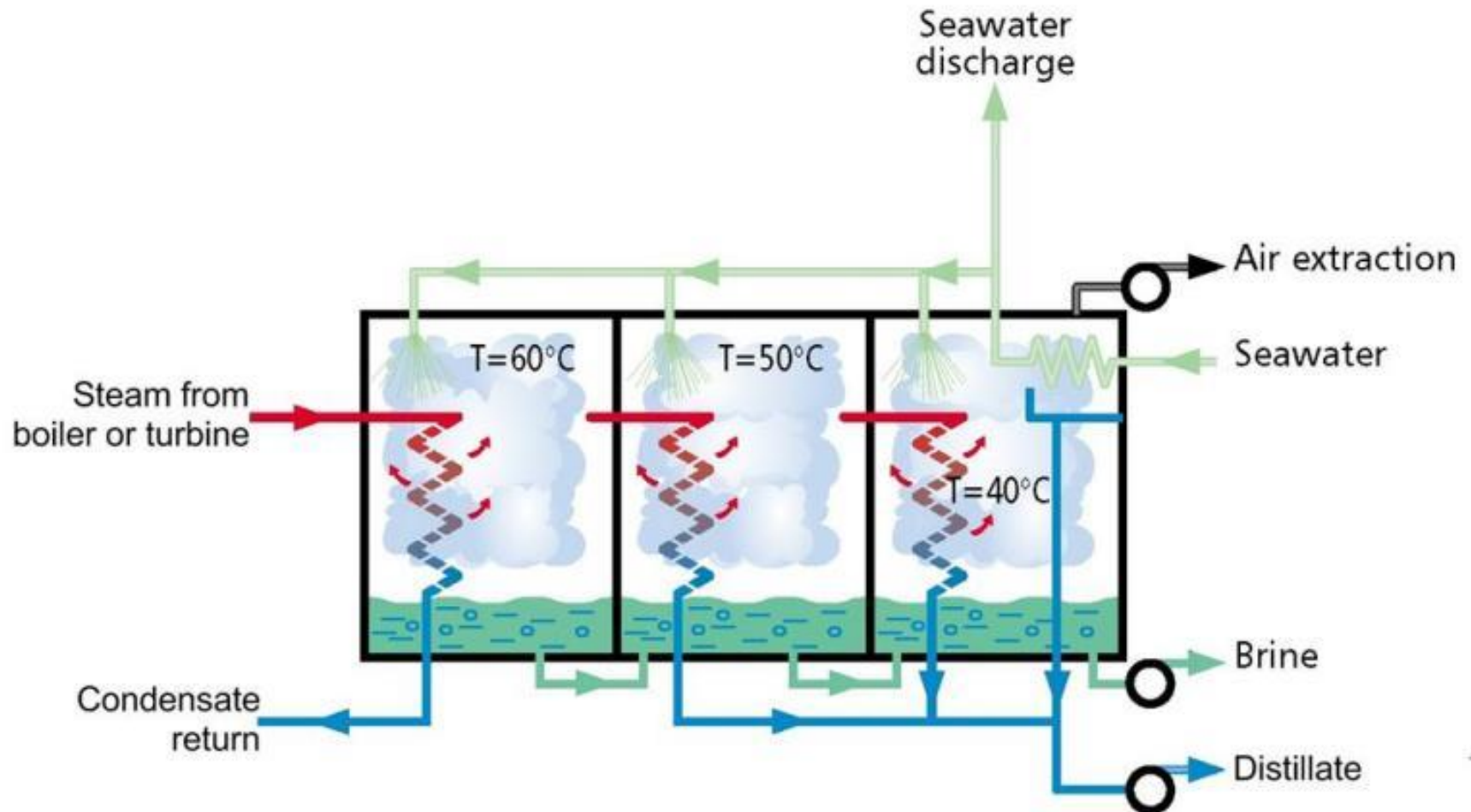
## **Solar Energy**

### **UNIT SIX**

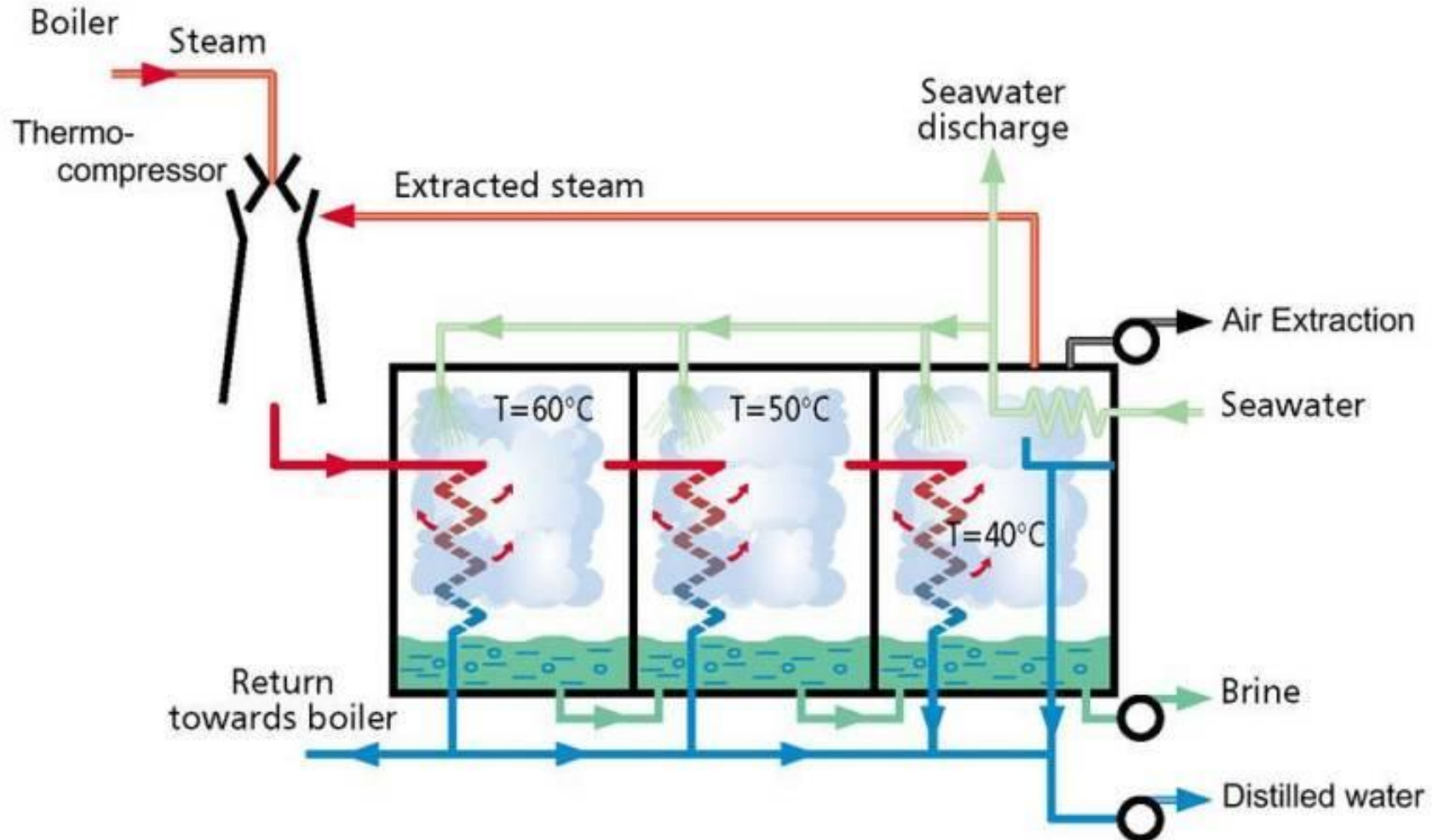
### **SOLAR ENERGY APPLICATIONS**

### ***THERMAL DESALINATION***

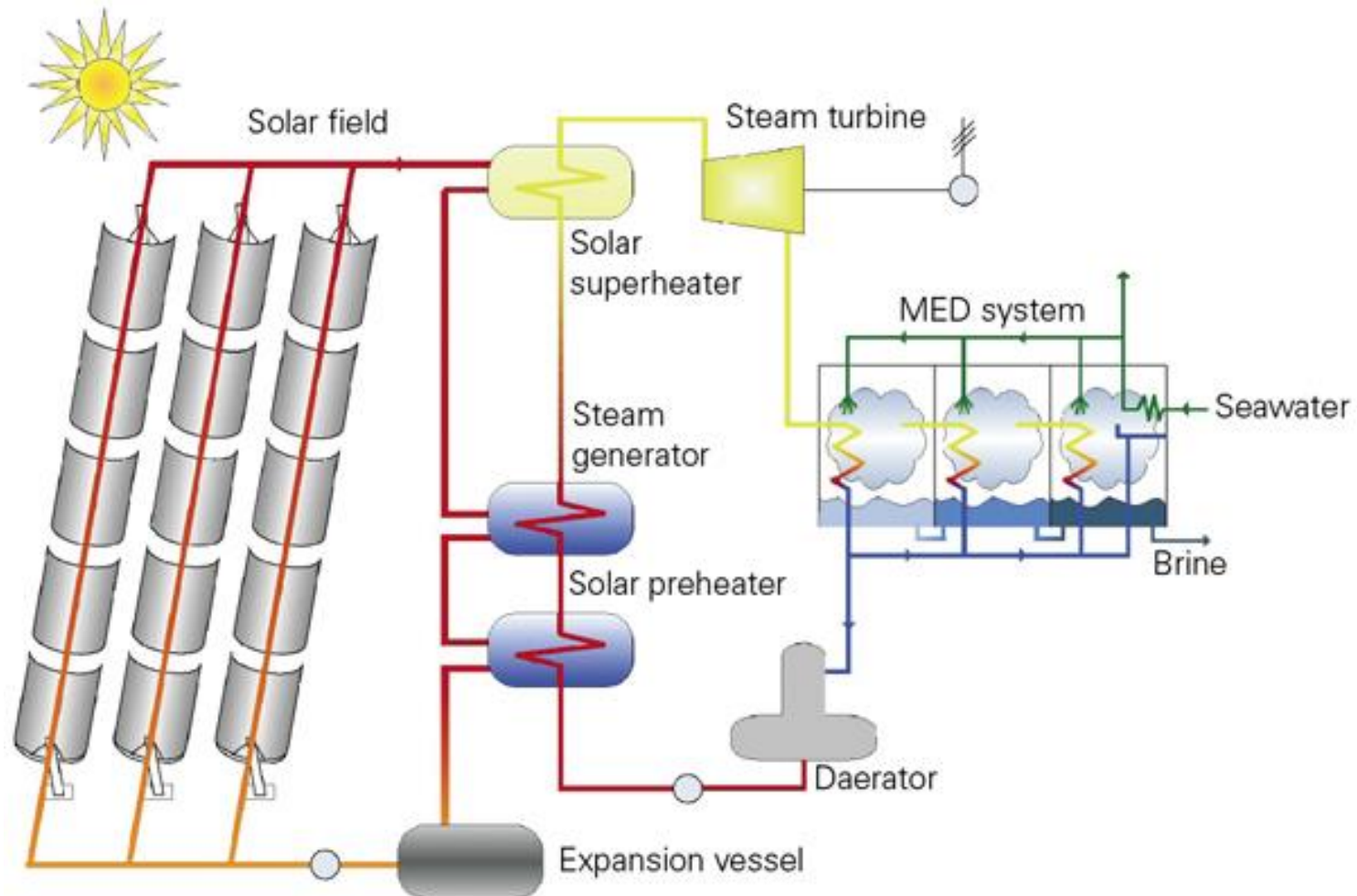
# Multiple Effect Distillation (MED)



# Multiple Effect Distillation with Thermal Vapor Compression (MED-TVC)

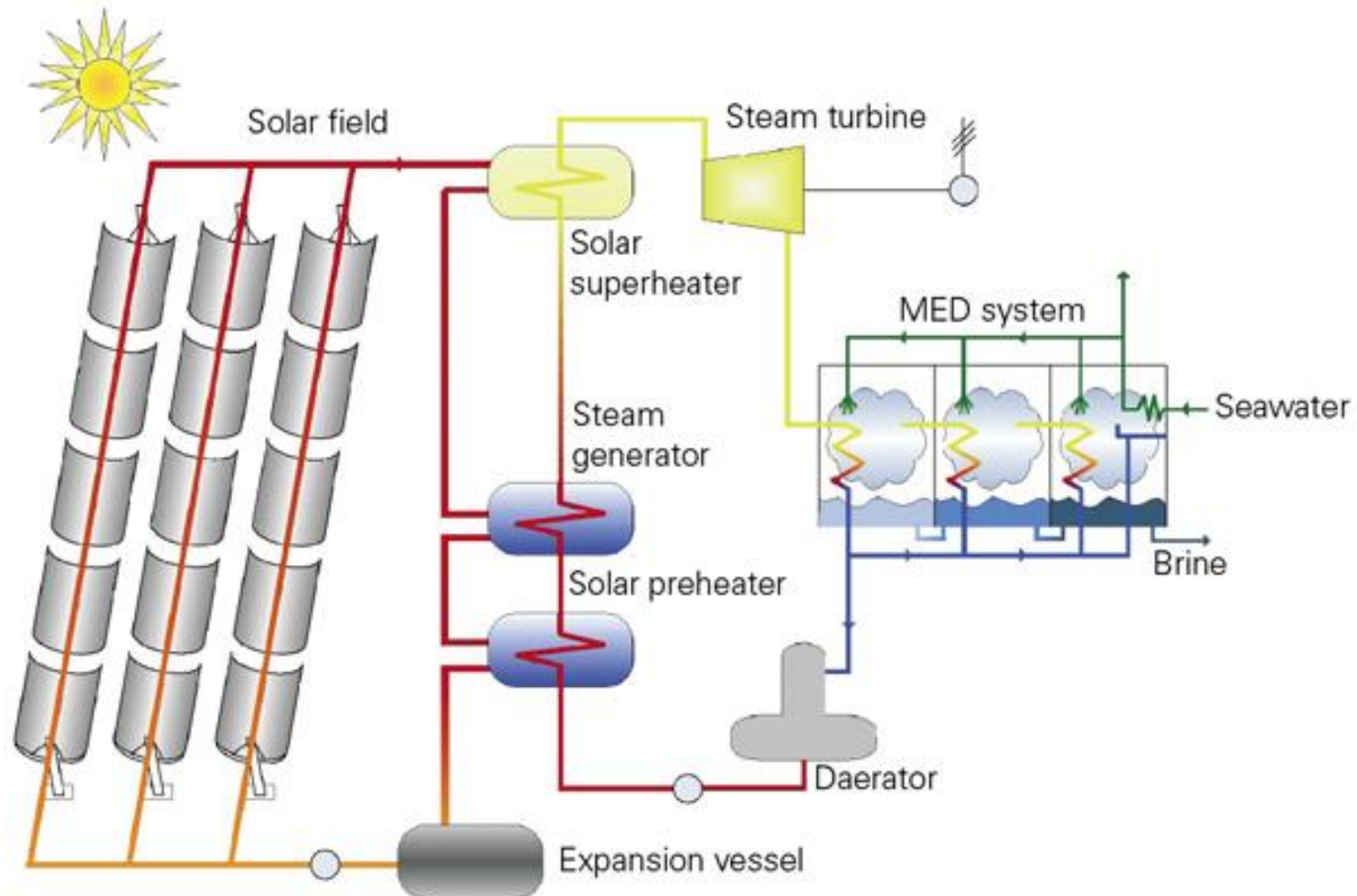


# Solar MED (With Power Generation)



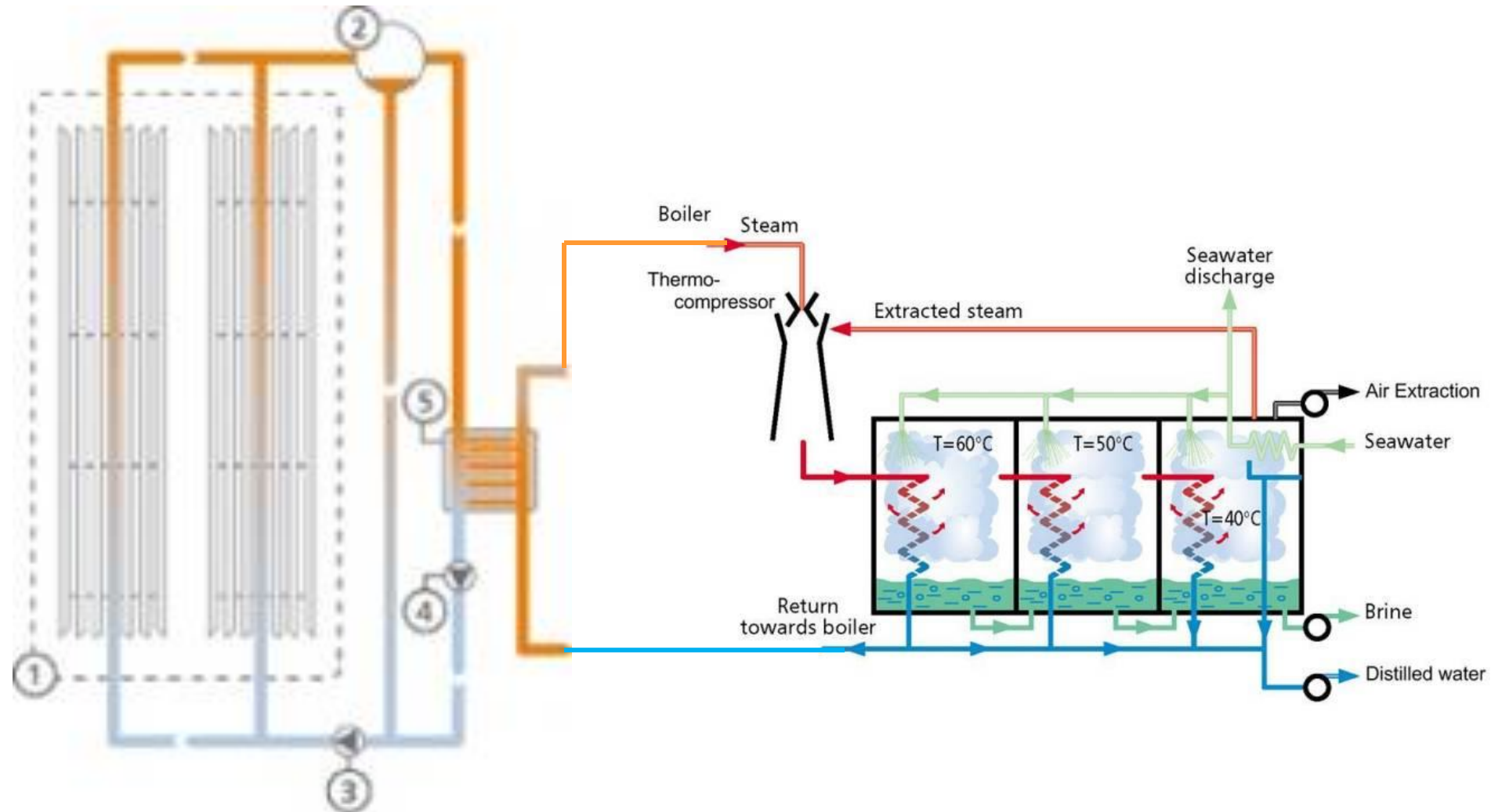
# Solar MED (With Power Generation)

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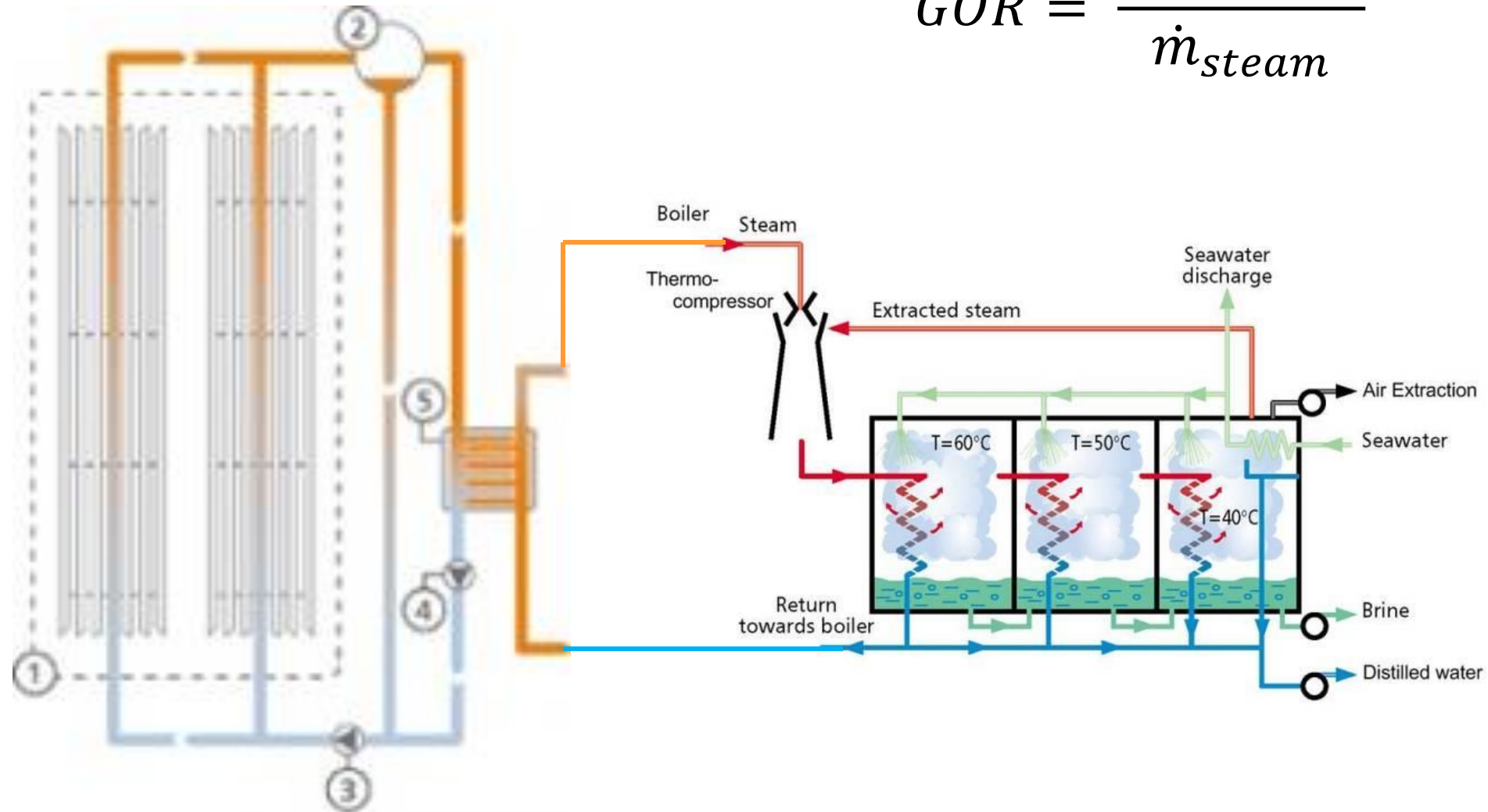


# Solar MED (Without Power Generation)



# Gain Output Ratio (GOR)

$$GOR = \frac{\dot{m}_{distillate}}{\dot{m}_{steam}}$$



# Gain Output Ratio (GOR)

$$GOR = \frac{\dot{m}_{distillate}}{\dot{m}_{steam}}$$

