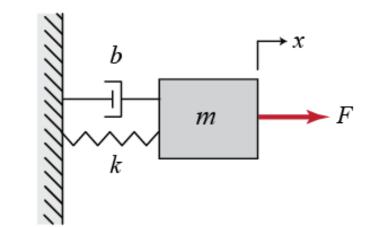
Exercise 1

Suppose we have a simple mass, spring, and damper problem. Find

- 1. The modeling equation of this system (F input, x output).
- 2. The transfer function.

Let M = 1 kg b = 10 N s/m k = 20 N/m F = 1 N

- 3. Find Open-Loop Step Response.
- 4. Design a proportional controller to improve the output (ω_n =17.89).
- 5. Design a proportional derivative controller to improve the output (ζ =0.56 and ω_n =17.89).



1. The modeling equation of this system

$$\sum f = M \frac{d^2 x}{dt^2}$$

 $M\ddot{x} = -b\dot{x} - kx + F \implies M\ddot{x} + b\dot{x} + kx = F$

2. The transfer function.

Taking the Laplace transform of the modeling equation, we get

$$Ms^{2}X(s) + b sX(s) + k X(s) = F(s) \implies G(s) = \frac{X(s)}{F(s)} = \frac{1}{Ms^{2} + b s + k}$$

Let M = 1 kg b = 10 N s/m k = 20 N/m F = 1 N
Plug these values into the above transfer function $G(s) = \frac{X(s)}{F(s)} = \frac{1}{s^{2} + 10 s + 20}$
B. Find Open-Loop Step Response.

Create a new m-file (Matlab) and run the following code:

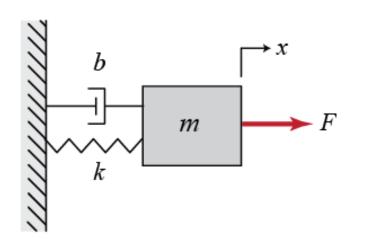
s = tf('s');

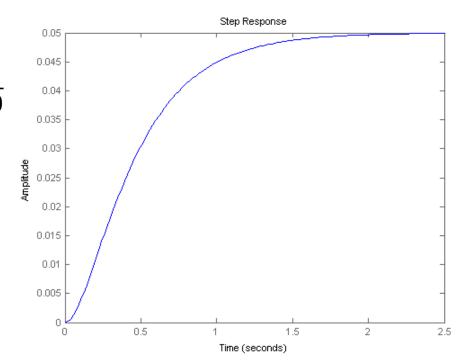
```
P = 1/(s^2 + 10*s + 20);
```

step(P)

3.

- the DC gain of the plant transfer function is 1/20, so 0.05 is the final value of the output to an unit step input.
- this corresponds to the steady-state error of 0.95, quite large indeed.
- the rise time is about one second,
- the settling time is about 1.5 seconds

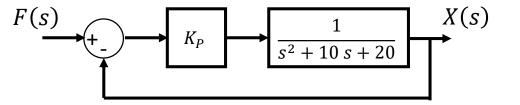




4. Proportional Control

The proportional controller (Kp) reduces the rise time, increases the overshoot, and reduces the steady-state error.

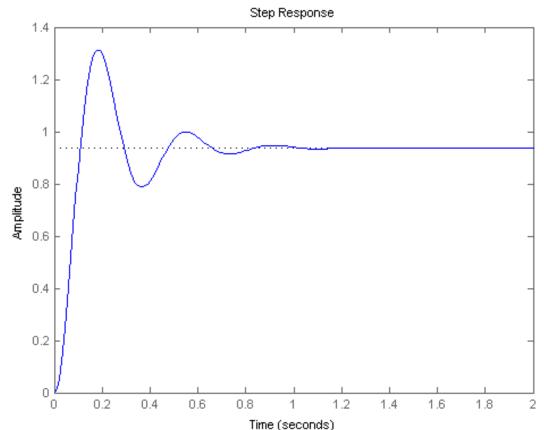
$$G(s) = \frac{X(s)}{F(s)} = \frac{K_P}{s^2 + 10 s + (20 + K_P)}$$
$$\omega_n^2 = 20 + K_P \Longrightarrow K_P = \omega_n^2 - 20 = 300$$



Let the proportional gain equal 300 and change the m-file to the following

Kp = 300; C = pid(Kp) T = feedback(C*P,1) t = 0:0.01:2; step(T,t)

The above plot shows that the proportional controller reduced both the rise time and the steady-state error, increased the overshoot, and decreased the settling time by small amount.



Proportional-Derivative Control

Now, let's take a look at a PD control. From the table shown above, we see that the derivative controller (Kd) reduces both the overshoot and the settling time. The closed-loop transfer function of the given system with a PD controller is:

$$20 + K_P = \omega_n^2 \implies K_P = \omega_n^2 - 20 = 300$$

10 + K_D = 2\zeta \omega_n \implies K_D = 2\zeta \omega_n - 20 = 2 (0.56)(17.89) - 10 \implies K_D = 10

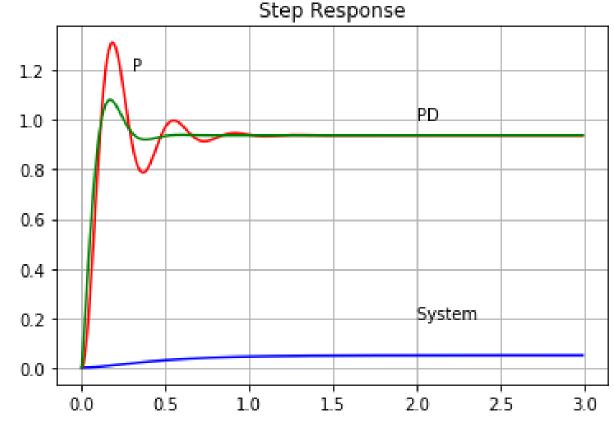
$$\frac{X(s)}{F(s)} = \frac{K_d s + K_p}{s^2 + (10 + K_d)s + (20 + K_p)}$$

Kp = 300; Kd = 10; C = pid(Kp,0,Kd)

T = feedback(C*P,1)

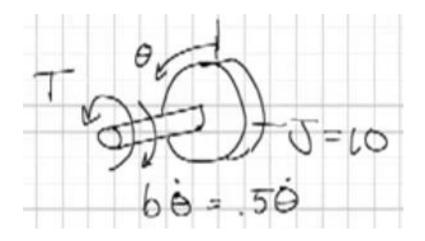
t = 0:0.01:2;

This plot shows that the derivative controller reduced both the overshoot and the settling time, and had a small effect on the rise time and the steady-state error.



Exercise 2

- The plant consists of rotating mass with inertia J and a viscous friction b, a torque T is applied to control the position of the mass.
- 1. Find the system model.
- 2. Find the transfer function.
- 3. Control the position θ using T to have ζ = 0.7 (Proportional controller)



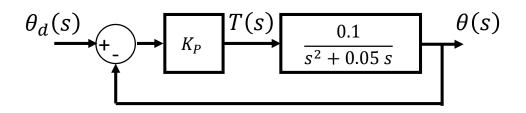
1. The modeling equation of this system

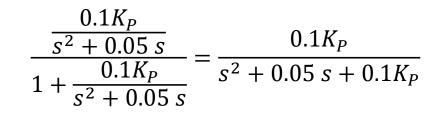
$$\sum Torques = J \frac{d^2\theta}{dt^2}$$
$$J\ddot{\theta} = T - b\dot{\theta} \Longrightarrow J\ddot{\theta} + b\dot{\theta} = T \implies \ddot{\theta} + 0.05 \dot{\theta} = 0.1 T$$

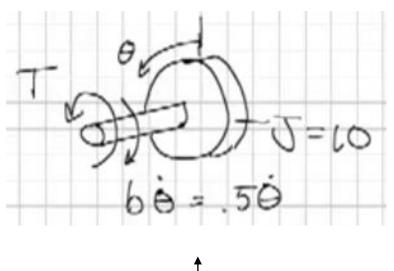
2. The transfer function

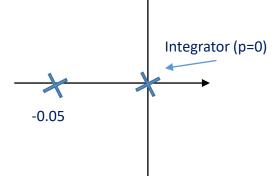
$$s^2\theta(s) + 0.05 \, S \, \theta(s) = 0.1 \, T(s) \Longrightarrow \frac{\theta(s)}{T(s)} = \frac{0.1}{s^2 + 0.05 \, s}$$

- 3. System control
- The open Loop Analysis
- $t_s = \frac{4}{\sigma} = \frac{4}{0.05} = 80 \ sec$ (very slow moving system)
- The close-Loop with Proportional controller Analysis









Given the characteristic equation we calculate the value of K_P to have the desired transient response

$$CLCE : s^{2} + 0.05 s + 0.1K_{P}$$

$$CLCE Desired : s^{2} + 2\zeta \omega_{n} s + \omega_{n}^{2}$$
The desired specifications: $\zeta = 0.7$

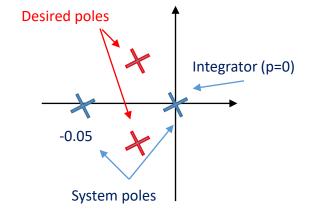
$$Matching = 2\zeta \omega_{n} = 0.05$$

$$0.1K_{P} = \omega_{n}^{2}$$

$$\omega_{n} = \frac{0.05}{2(0.7)} = 0.0357 \ rad/sec$$

$$K_{P} = 10 \ (0.0357)^{2} = 0.0127$$

The desired poles: $P_{1,2} = -\zeta \omega_n \pm j \omega_n \sqrt{1 - \zeta^2} = -0.025 \pm j 0.025$



The plant G(s) is given $G(s) = \frac{10}{s^2 - 3s + 2}$, design a PD controller (using pole placement) to have the desired system response: $\xi = 0.5$, and $T_s = 1s$

1. Find the desired poles and give the PD controller transfer function $G_c(s)$ (general form). 2. Find the closed loop transfer function CLTF (with PD controller).

- 3. Find the closed loop characteristic equation CLCE.
- 4. Find the gains K_D and K_P of the PD controller.

1. desired poles at :
$$T_s = \frac{4}{\xi \omega_n} \rightarrow \omega_n = \frac{4}{\xi T_s} = 8 rad/sec$$

$$s_{1,2} = -\xi \omega_n \pm \omega_n \sqrt{1 - \xi^2} = -4 \pm j \ 6.93$$

PD controller transfer function: $G_c(s) = K_P + K_D s$

2. CLTF:
$$T(s) = \frac{G_c(s)G(s)}{1+G_c(s)G(s)} = \frac{10 (K_P + K_D s)}{s^2 + (10 K_D - 3)s + 10 K_P + 2}$$

3. CLCE: $s^2 + (10 K_D - 3)s + 10 K_P + 2 = 0$

4. Desired CLCE: $s^2 + 2 \boldsymbol{\xi} \boldsymbol{\omega}_n s + \omega_n^2 = 0$

PD controller Gains:
$$K_P = \frac{\omega_n^2 - 2}{10}$$
 and $K_D = \frac{2 \xi \omega_n + 3}{10}$
 $K_P = 6.2$ and $K_D = 1.1$

Exercise 3

Use a PI controller to control the system $G(s) = \frac{2}{s+4}$ to meet the specifications $\zeta = 0.7$ and $t_s < 0.5$ sec.

The system is type zero, it has no integrator. In order to have no steady-state error we need to add an integrator to make the system type one.

The open Loop Analysis: the system pole at p = -3.

The open Loop Analysis: the system pole at p= - 3.

$$\zeta = 0.7.$$

$$t_s = \frac{4}{\sigma} < 1 \quad \implies \sigma = \zeta \omega_n > 4 \quad \implies \omega_n > \frac{4}{\zeta} = \frac{4}{0.7} \quad \implies \omega_n > 5.71 \quad \implies \omega_n = 6$$
The controller design: PI controller $y_d(\underline{s}) \quad \xleftarrow{K_r + \frac{K_I}{s}} \quad \boxed{\frac{1}{s+3}} \quad \boxed{\frac{1}{s+3}} \quad \boxed{y(s)}$
The PI controller can be written as:
$$K_P + \frac{K_I}{s} = K_P \frac{(s + \frac{K_I}{K_P})}{\frac{Gain}{s}} \quad \underbrace{Zero = K_{IP}}{\frac{S}{Gain}}$$

$$CLCE : s(s+3) + K_P \left(s + \frac{K_I}{K_P}\right) = 0 \quad \implies s^2 + (2 + K_P) s + K_I = 0$$
Desired CLCE : $s^2 + 2\zeta \omega_n s + \omega_n^2 = s^2 + 2(0.7)(6)s + (6)^2 = s^2 + 8.4 s + 36 = 0$

$$\implies 2 + K_P = 8.4$$

$$K_I = 36$$