



# ME 476

## Solar Energy

### UNIT FOUR

# SOLAR COLLECTORS

## Flat Plate Collectors

- What are flat plate collectors?
- Types of flat plate collectors
- Applications of flat plate collectors
- Thermal analysis of flat plate collectors

# What Are Flat Plate Collectors?

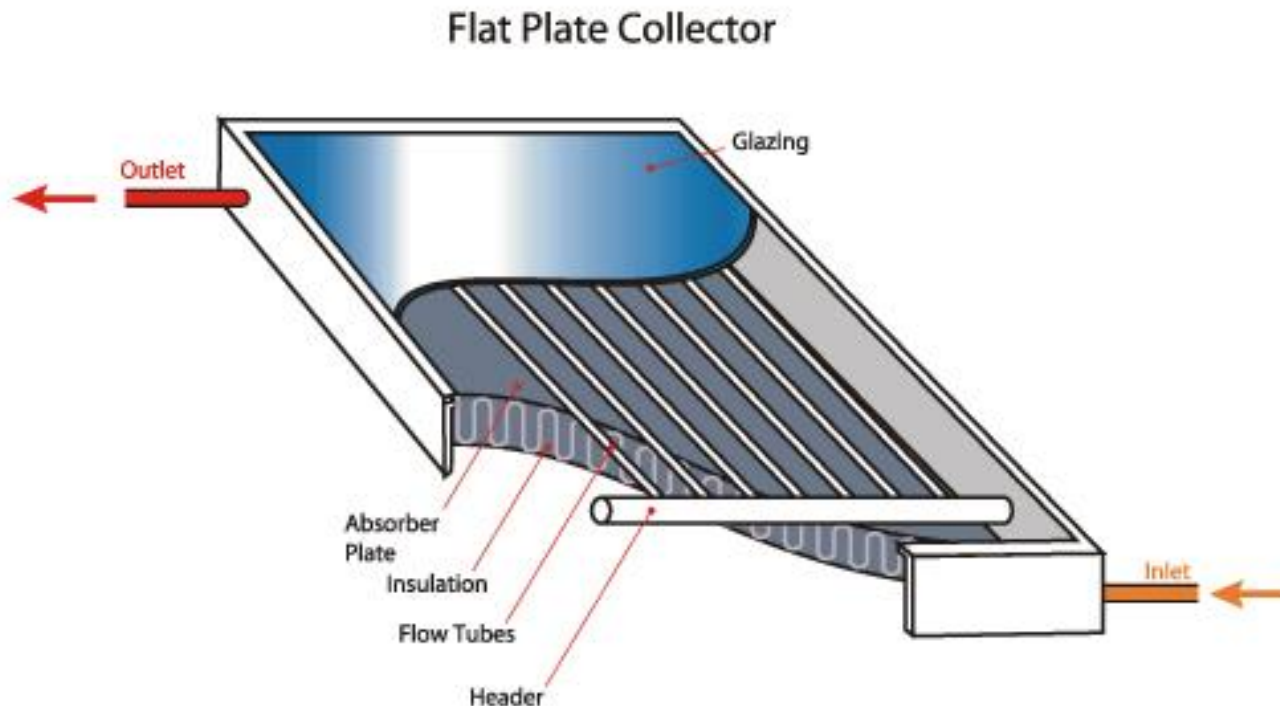
- A flat plate collector is a heat exchanger that uses solar irradiation to heat a working fluid.
- The working fluid is usually liquid or air.
- The collector is a black surface that is placed at a convenient path of the sun.
- In flat plate collectors there is no optical concentration of sunlight and they are generally stationary.
- The outlet temperature capability is below 100 °C



# What Are Flat Plate Collectors?

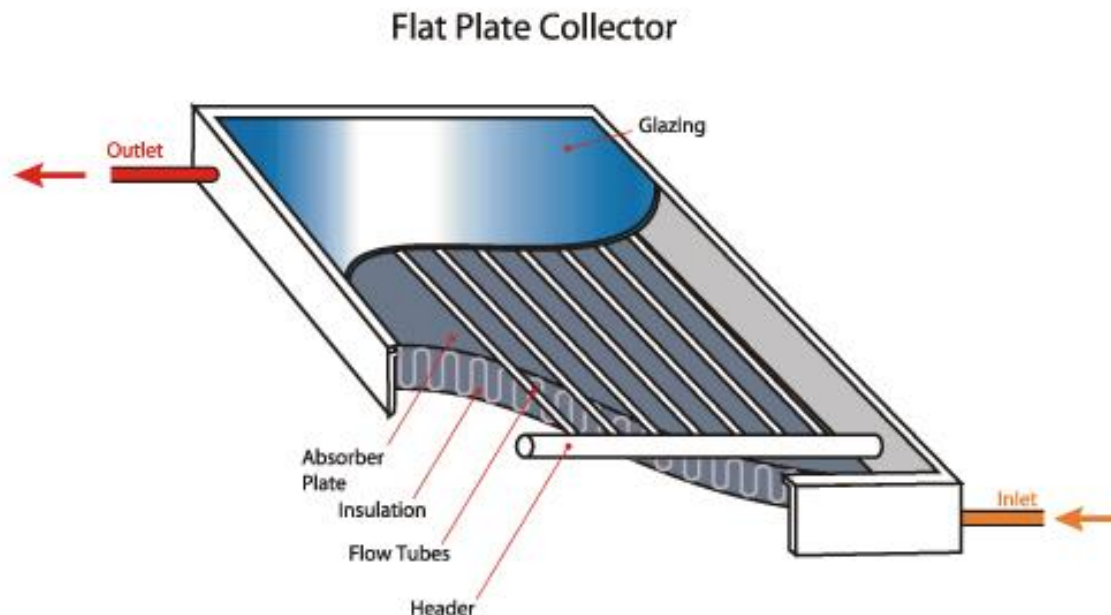
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- A typical flat plate collector is a metal box with a glass or plastic cover (called glazing) on top and a dark-colored absorber plate on the bottom.
- The sides and bottom of the collector are usually insulated to minimize heat loss.



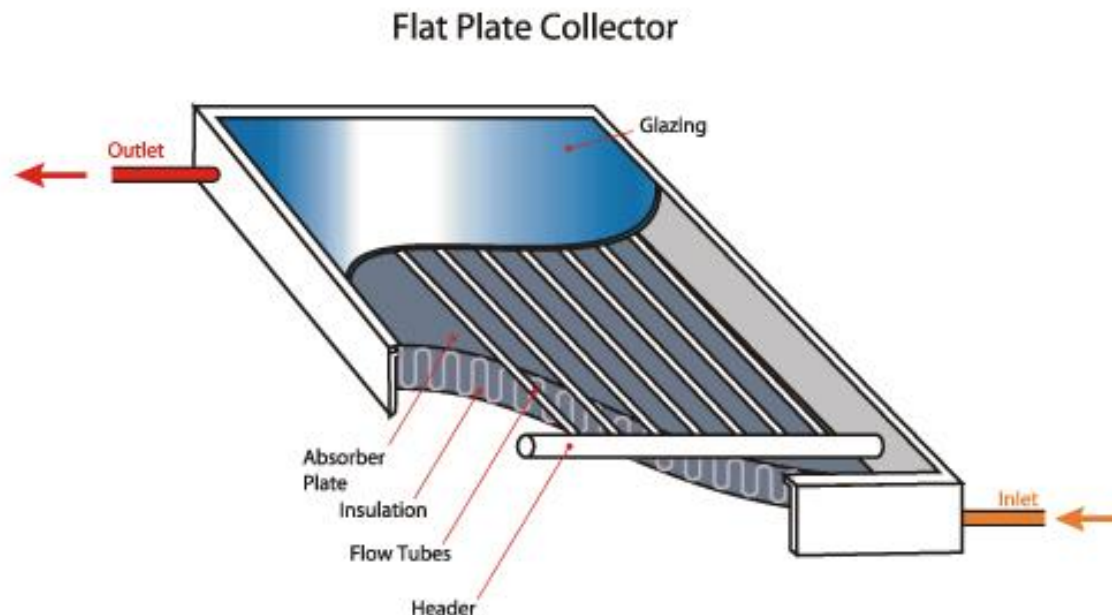
## ABSORBER PLATE

- The plate is usually made of copper, steel, or plastic.
- The surface is covered with a black material of high absorptance.
- A selective coating can be used to maximize the absorptance of solar energy and minimizes the radiation emitted by plate.



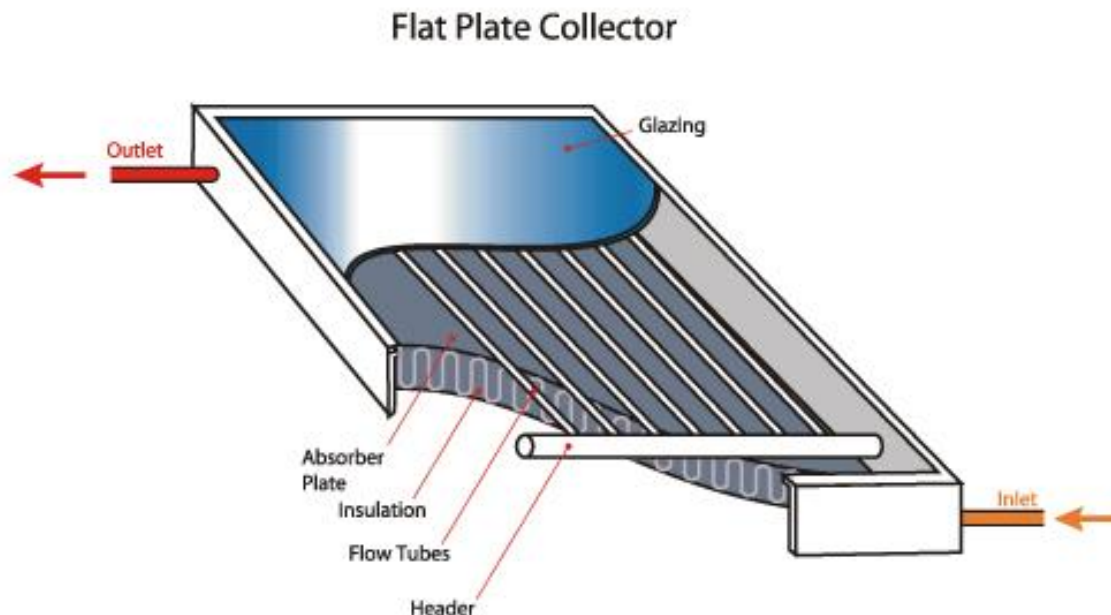
## FLOW PASSAGES

- The flow passages carry the working fluid through the collector.
- If the working fluid is a liquid, the flow passage is usually a tube that is attached to, or is a part of absorber plate.
- If the working fluid is air, the flow passages can have different configurations.



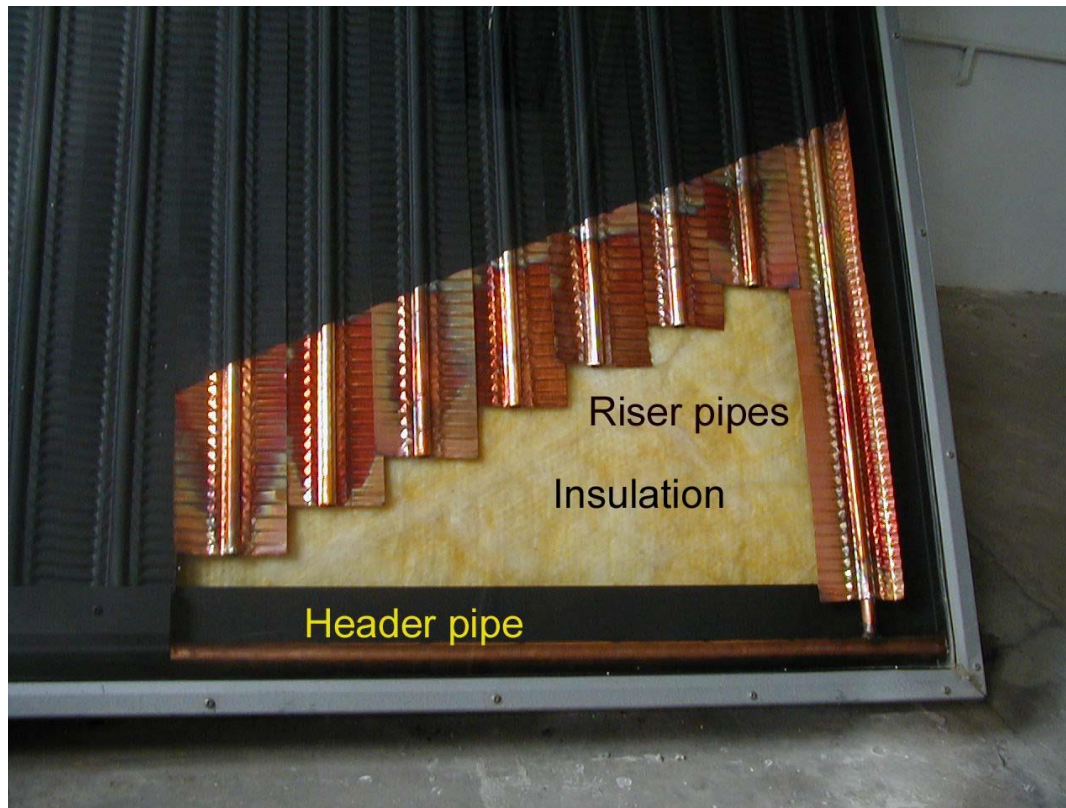
## COVER PLATE (GLAZING)

- To reduce convective and radiative heat losses from the absorber, one or two transparent covers (glazing) are generally placed above the absorber plate.
- They usually be made from glass or plastic.



## INSULATION

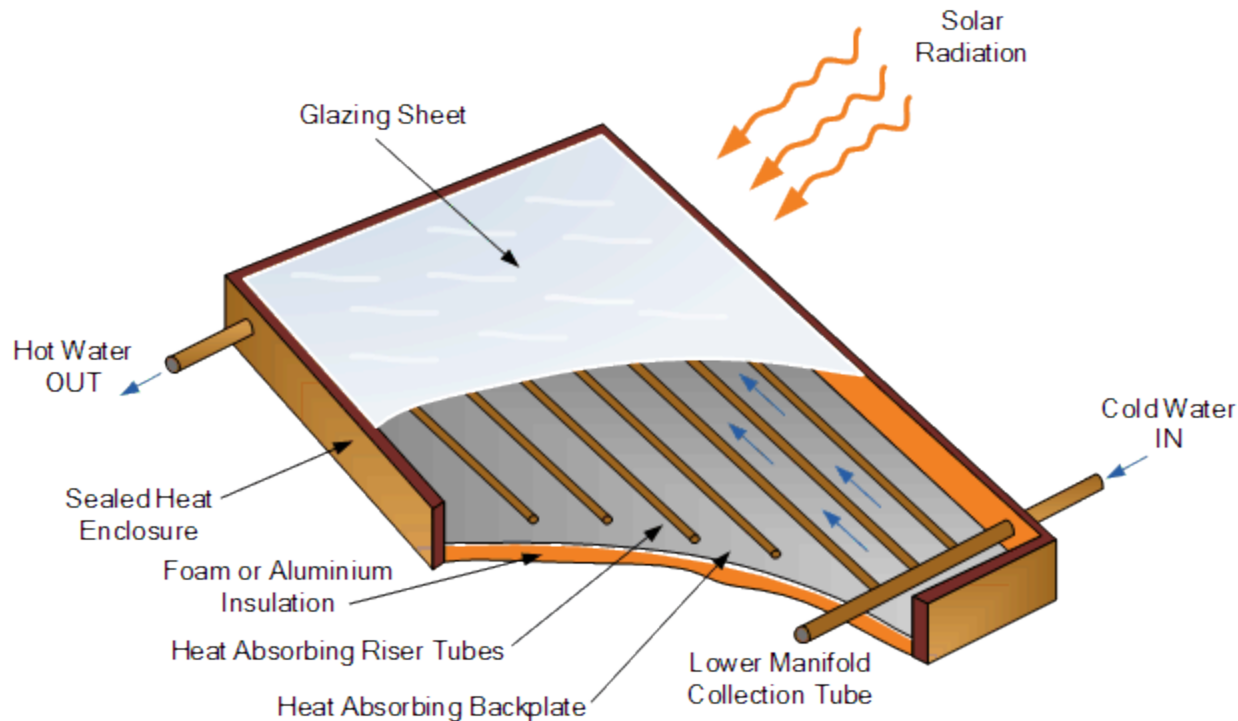
- These are some materials such as fiberglass and they are placed at the back and sides of the collector to reduce heat losses.





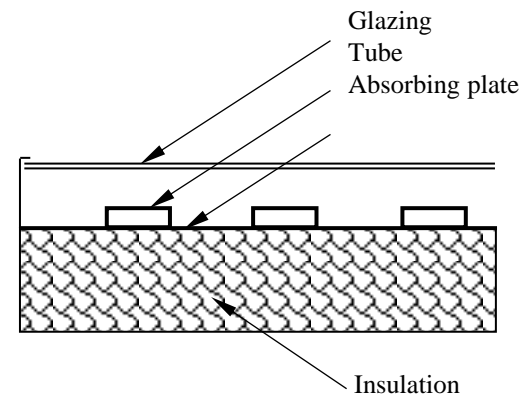
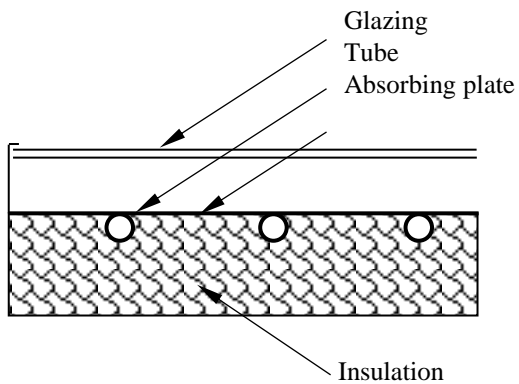
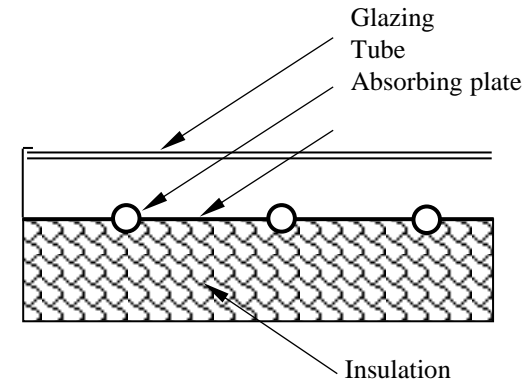
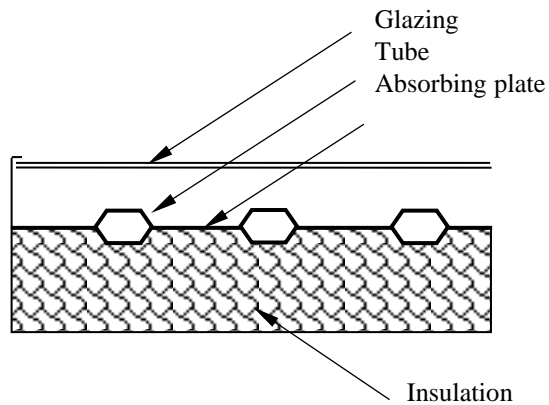
## ENCLOSURE

- A box that encloses the collector to:
  - **Hold all the components together**
  - **Protect them from weather**
  - **Facilitate installation on a roof or appropriate frame.**

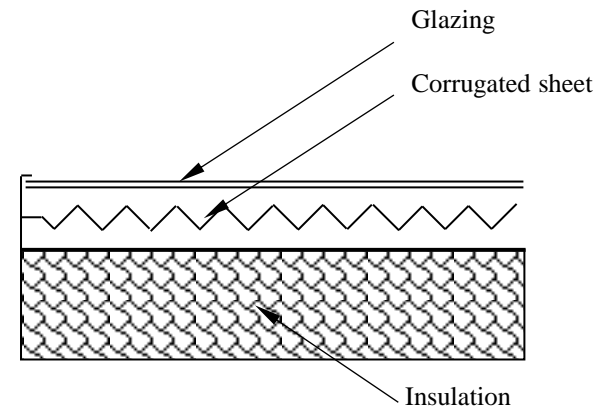
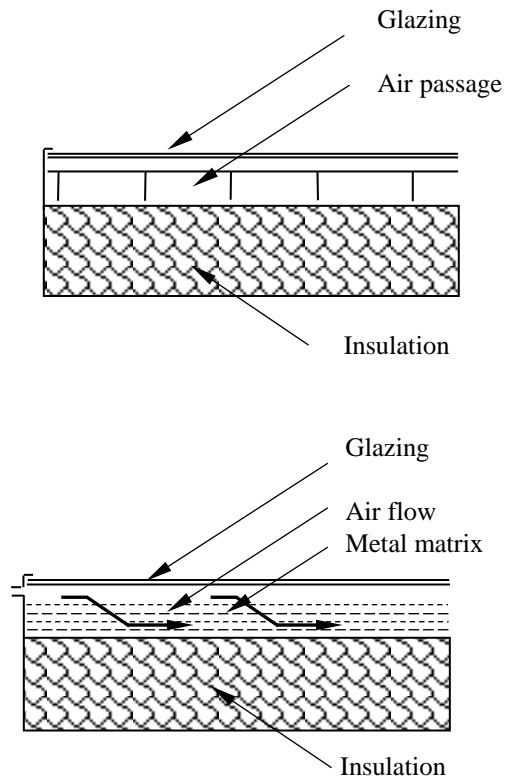


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## WATER SYSTEMS

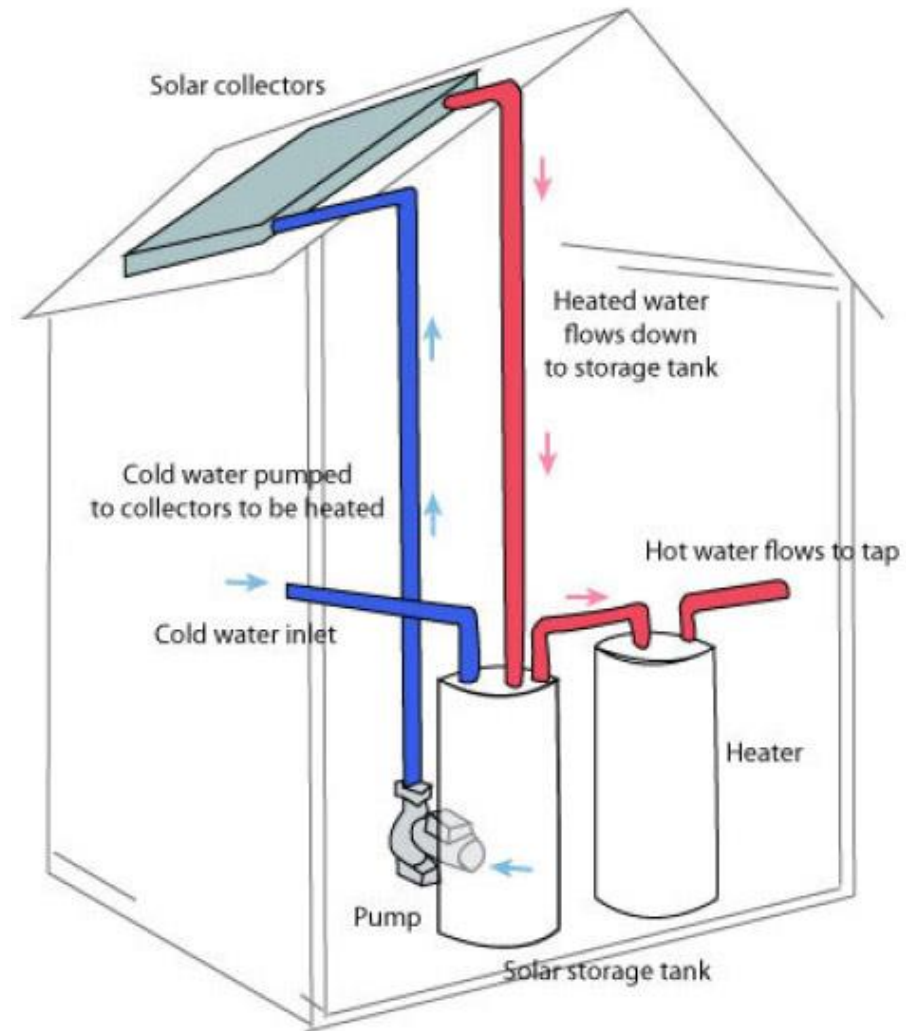
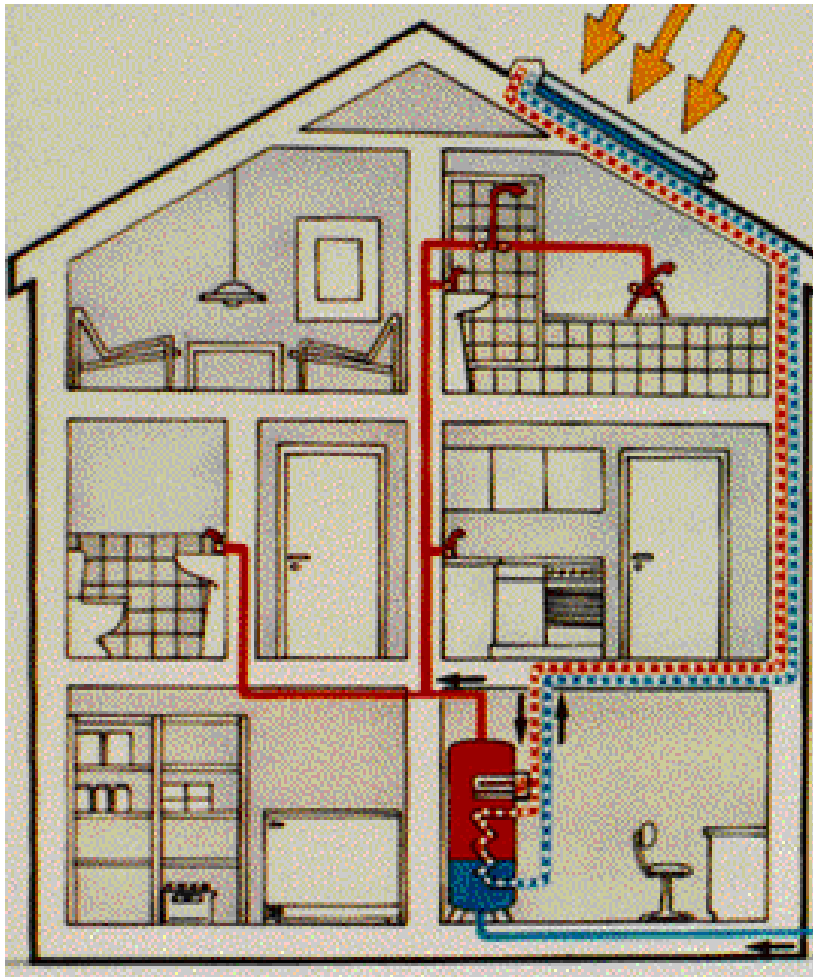


## AIR SYSTEMS

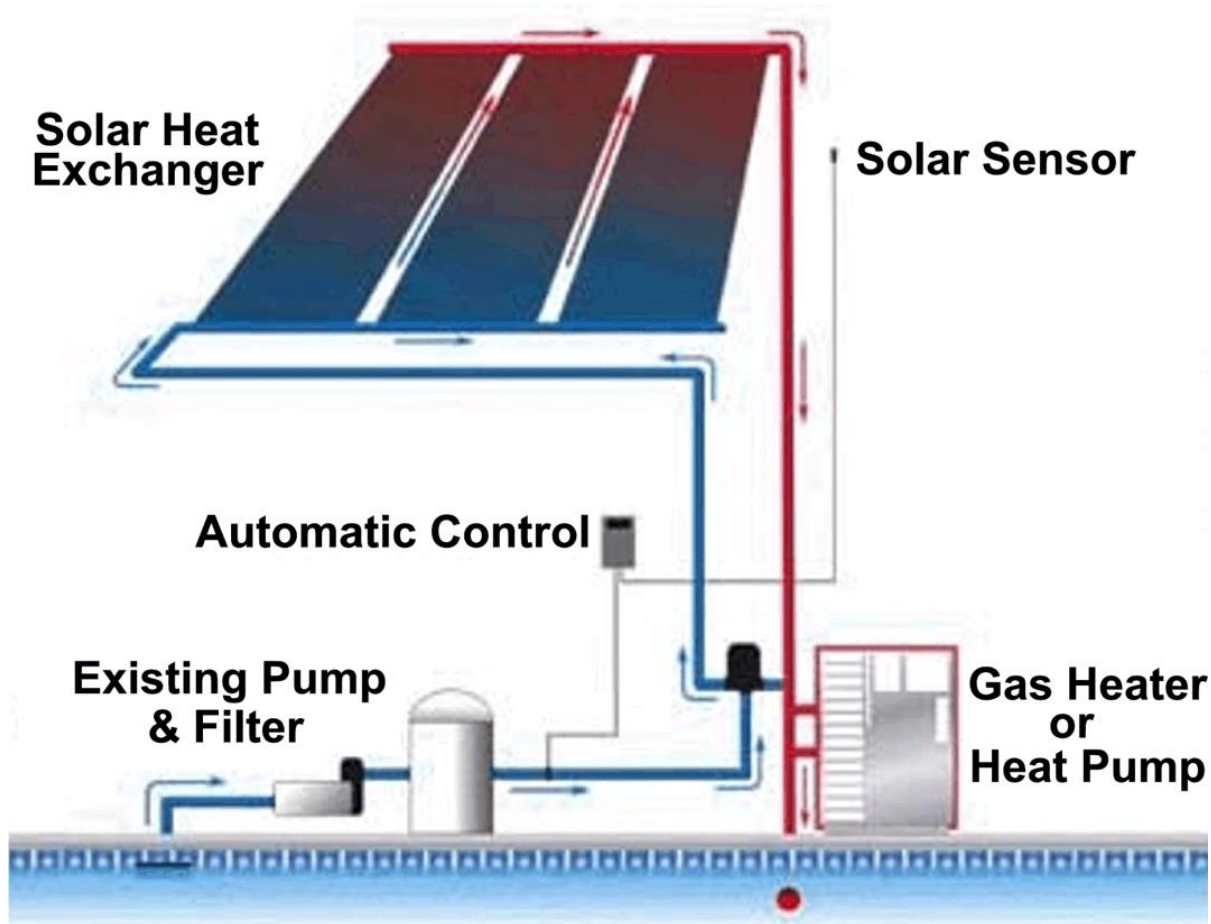


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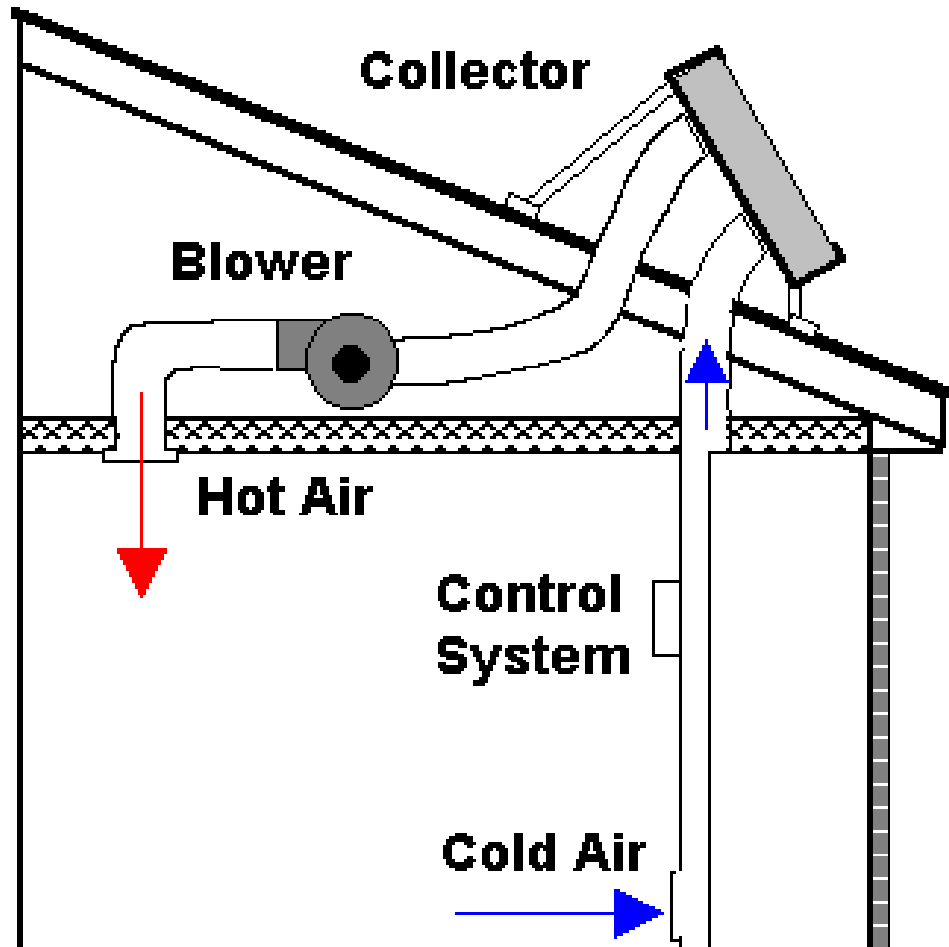
## DOMESTIC HOT WATER



## WATER HEATING



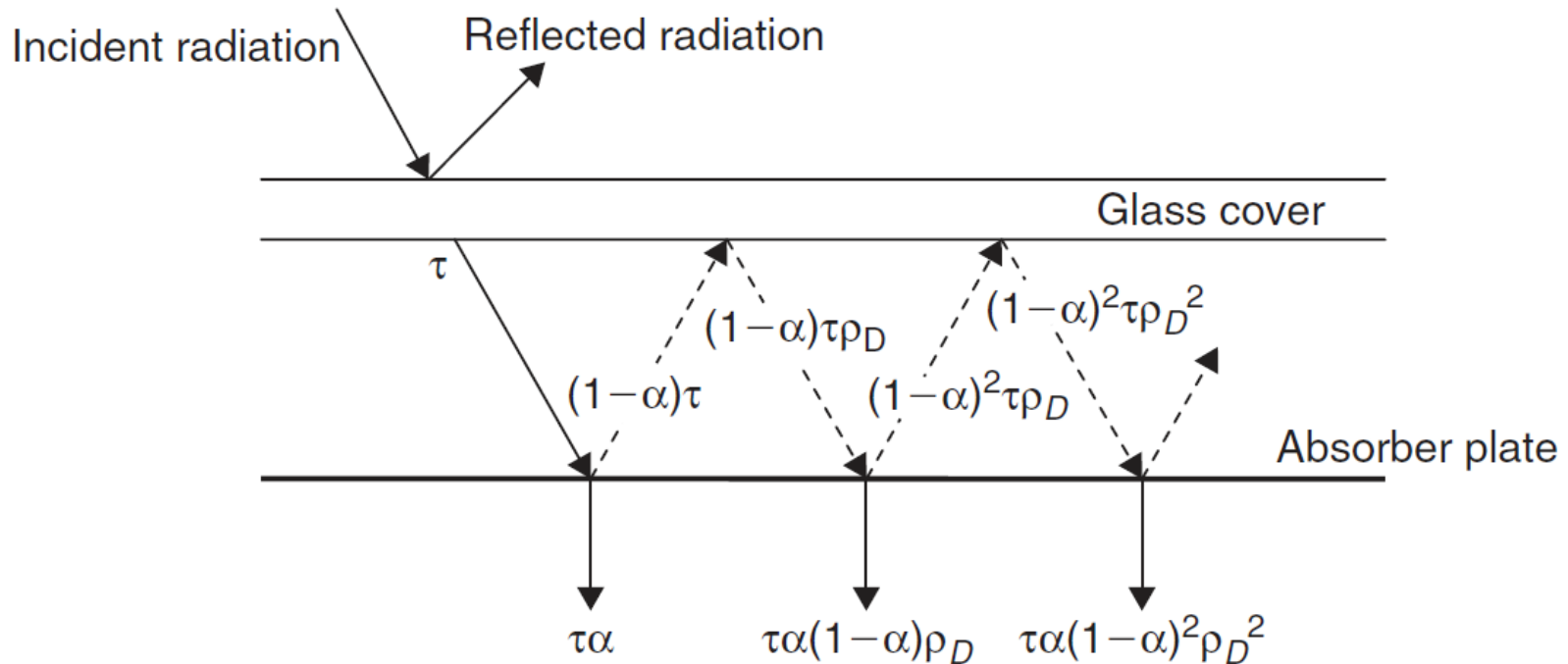
## SPACE HEATING





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- The irradiation incident on a collector ( $G_t$ ) is not all absorbed.
- Once the irradiation penetrates the glass cover, part of it is absorbed by the collector, but another part is reflected back diffusely to the glass cover.
- The glass cover then reflects diffusely to the absorber, and so on.



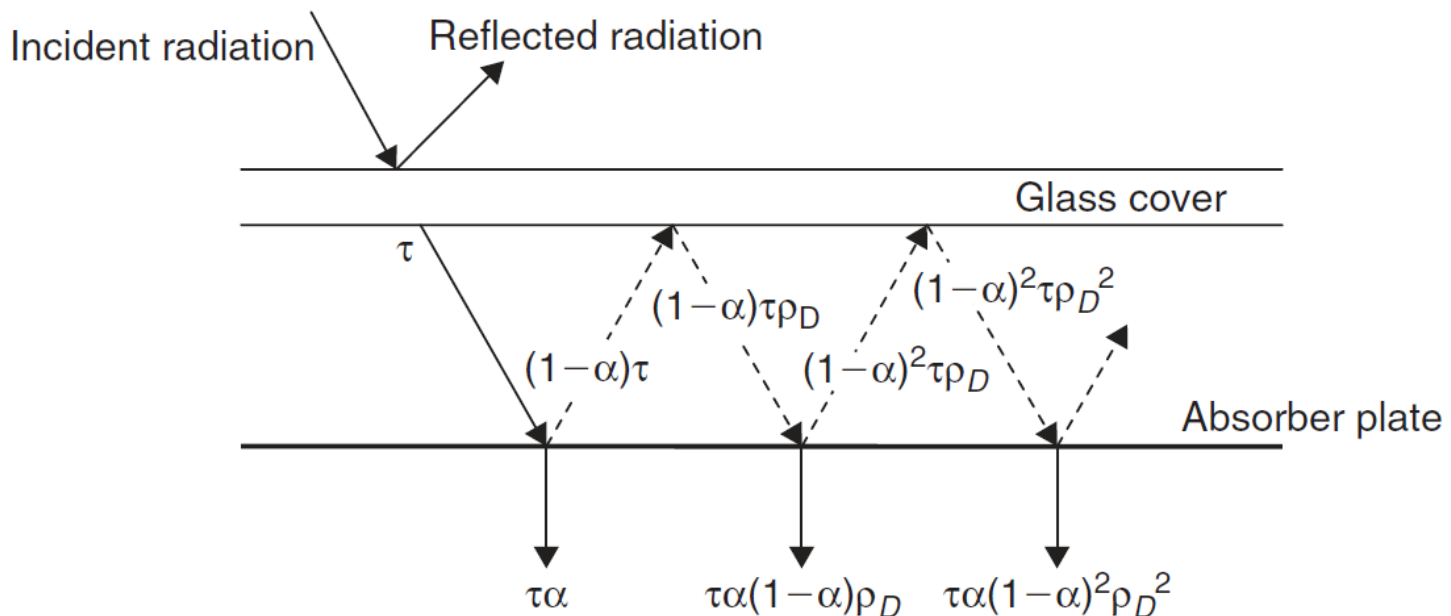
# Energy Absorbed by a Flat Plate Collector

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- The net energy absorbed by the collector can be expressed in terms of a quantity called  $(\tau\alpha)$ .
- Theoretically, the net energy absorbed by the collector per unit area is:

$$S = G_t (\tau\alpha)_{av}$$

Where  $(\tau\alpha)_{av}$  is the average value of  $(\tau\alpha)$



- The useful energy gain of a flat plate collector is given by:

$$Q_u = S \times A_c - Q_{\text{loss}}$$

- $Q_{\text{loss}}$  can be due to energy loss through:

- **Top of the collector**
- **Bottom of collector**
- **Edges of collector**

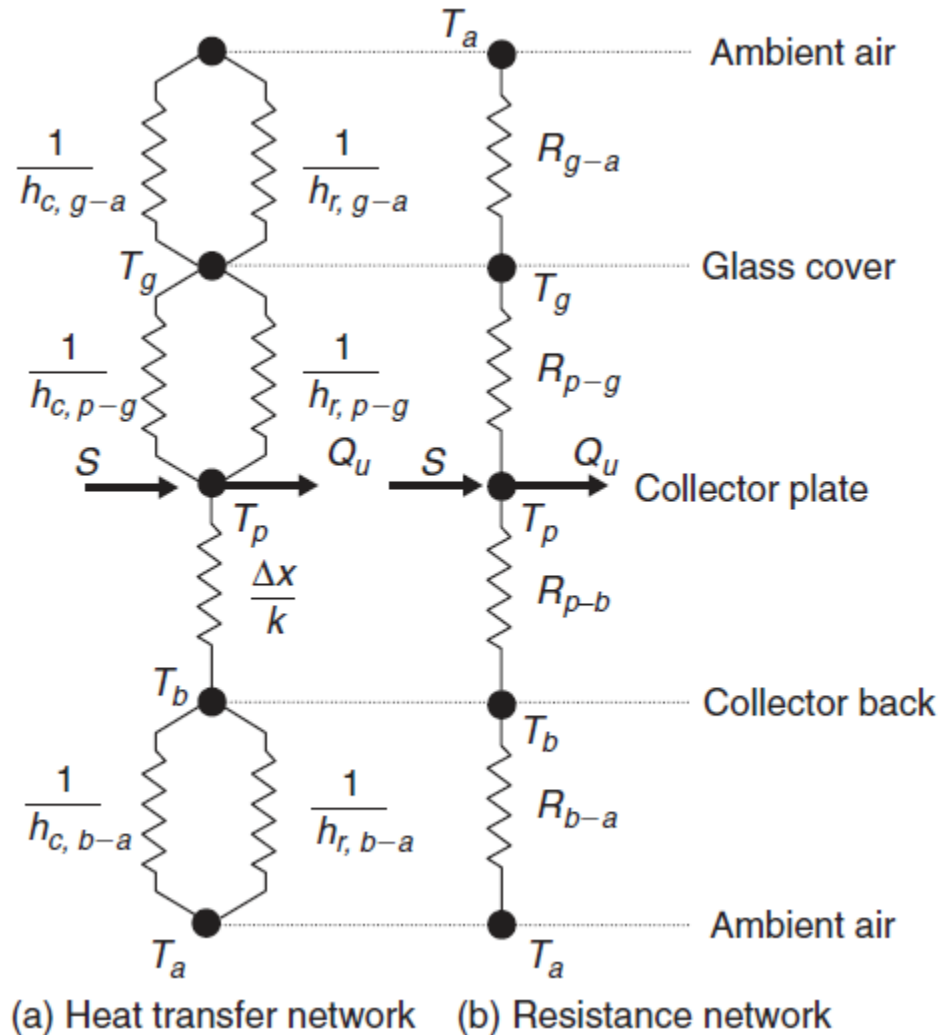
- $Q_{\text{loss}}$  can be expressed as:  $Q_{\text{loss}} = U_L A_c (T_p - T_a)$

where,

- $T_p$ : mean temperature of the absorber plate
- $T_a$ : ambient temperature
- $U_L$ : overall heat transfer coefficient based on collector area

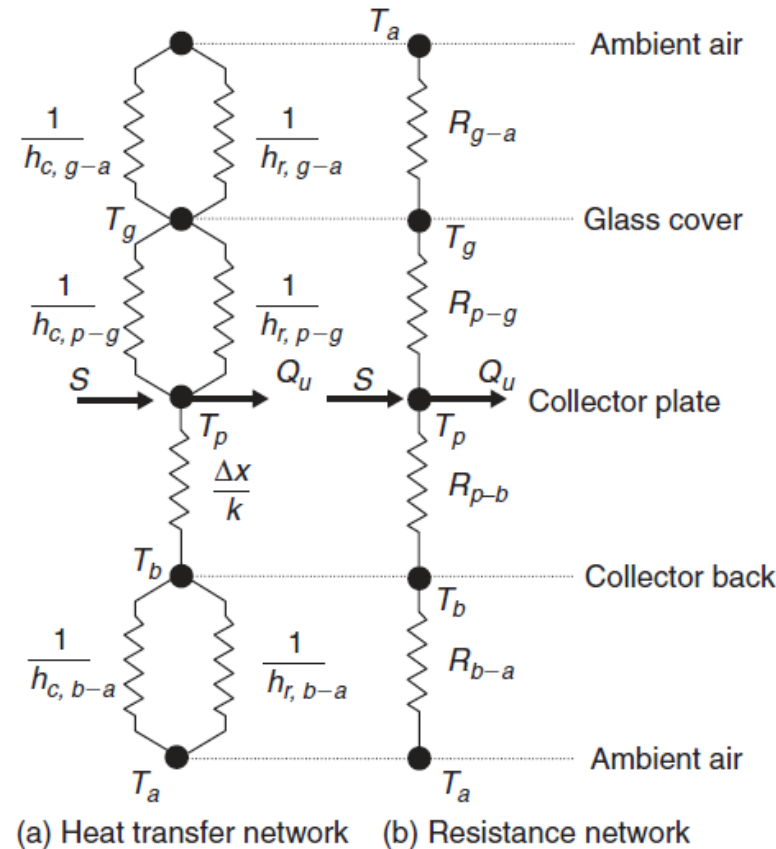
# Energy Loss of a Flat Plate Collector

- $U_L$  consists of  $U_t$  (top),  $U_b$  (bottom), and  $U_e$  (edges)



$U_t$  involves the calculation of:

- $h_{c,p-g}$ : convective heat transfer coefficient between plate and glazing
- $h_{r,p-g}$ : radiative heat transfer coefficient between plate and glazing
- $h_{c,g-a}$ : convective heat transfer coefficient between glazing and ambient air
- $h_{r,g-a}$ : radiative heat transfer coefficient between glazing and ambient air



- Calculations of all the components of  $U_t$  is complicated.

- A relatively simple alternative formula can be used:

$$U_t = \frac{1}{N_g} \frac{1}{\frac{C}{T_p} \left[ \frac{T_p - T_a}{N_g + f} \right]^{0.33} + \frac{1}{h_w} + \frac{\sigma(T_p^2 + T_a^2)(T_p + T_a)}{\frac{1}{\varepsilon_p + 0.05N_g(1 - \varepsilon_p)} + \frac{2N_g + f - 1}{\varepsilon_g} - N_g}}$$

$$f = (1 - 0.04h_w + 0.0005h_w^2)(1 + 0.091N_g)$$

$$C = 365.9 (1 - 0.00883 \alpha + 0.0001298 \alpha^2)$$

$$h_w = \frac{8.6V^{0.6}}{L^{0.4}}$$

$N_g$  = number of glass covers

$T_p$  = absorber plate temperature

$T_a$  = ambient temperature

$\sigma$  = Stefan-Boltzmann constant

$\varepsilon_p$  = emittance of absorber plate

$\varepsilon_g$  = emittance of glazing

$\alpha$  = Tilt angle

$V$  = Wind velocity

$L$  = Collector length



- $U_b$  can be found from:

$$U_b = \frac{1}{\frac{t_b}{k_b} + \frac{1}{h_{c,b-a}}}$$

Where,

$t_b$  = thickness of back insulation (m).

$k_b$  = conductivity of back insulation (W/m-K).

$h_{c,b-a}$  = convection heat loss coefficient from back to ambient (W/m<sup>2</sup>-K).

- $U_e$  can be found from:

$$U_e = \frac{1}{\frac{t_e}{k_e} + \frac{1}{h_{c,e-a}}}$$

Where,

$t_e$  = thickness of edge insulation (m).

$k_e$  = conductivity of edge insulation (W/m-K).

$h_{c,e-a}$  = convection heat loss coefficient from edge to ambient (W/m<sup>2</sup>-K).

- The useful energy gain of a flat plate collector is given by:

$$Q_u = S \times A_c - Q_{\text{loss}}$$

- Expanding all terms,

$$Q_u = A_c [G_t(\tau\alpha) - U_L(T_p - T_a)]$$

- The useful energy gained by the collector is transferred completely to the working fluid. Therefore,

$$Q_u = A_c [G_t(\tau\alpha) - U_L(T_p - T_a)] = \dot{m}c_p[T_o - T_i]$$

Where,

- $T_i$ : fluid inlet temperature
- $T_o$ : fluid outlet temperature

$$Q_u = A_c [G_t(\tau\alpha) - U_L(T_p - T_a)] = \dot{m}c_p[T_o - T_i]$$

- Calculating  $T_p$  accurately is difficult.
- It is more convenient to express  $Q_u$  in terms of the fluid temperatures.
- A useful definition is the heat removal factor ( $F_R$ ):

$$F_R = \frac{\text{Actual output}}{\text{Output for plate temperature = Fluid inlet temperature}}$$

- By using the heat removal factor ( $F_R$ ), the useful energy gain equation becomes:

$$Q_u = A_c F_R [G_t (\tau \alpha) - U_L (T_i - T_a)]$$

- $F_R$  depends on many factors, and it can be found analytically.
- $F_R$  can also be found experimentally.

- Collector efficiency ( $\eta$ ) is defined as the:
- $\eta = \text{useful energy gain} / \text{irradiation incident on the collector}$

$$\eta = \frac{Q_u}{G_t A_c}$$

- This equation can be expressed in terms of  $F_R$ :

$$\eta = F_R \left[ (\tau\alpha) - \frac{U_L(T_i - T_a)}{G_t} \right]$$

- The efficiency equation can be rearranged as follows:

$$\eta = F_R(\tau\alpha) - F_R U_L \frac{T_i - T_a}{G_t}$$

- If changes in  $F_R$  and  $U_L$  are small, the equation above represents a straight line, where:
  - $(T_i - T_a) / G_t$  is the independent variable
  - $\eta$  is the dependent variable
  - $F_R(\tau\alpha)$  is the intercept
  - $(- F_R U_L)$  is the slope

$$\eta = \underbrace{F_R(\tau\alpha)}_{\text{Intercept}} - \underbrace{F_R U_L}_{\text{Slope}} \frac{T_i - T_a}{G_t}$$

